



**UNIVERSITI TUN HUSSEIN ONN MALAYSIA**

**FINAL EXAMINATION**

**SEMESTER I**

**SESSION 2017/2018**

COURSE NAME : AIR CONDITIONING SYSTEM DESIGN  
COURSE CODE : BDE40103  
PROGRAMME : BDD  
EXAMINATION DATE : DECEMBER 2017 / JANUARY 2018  
DURATION : 3 HOURS  
INSTRUCTION : **PART A: ANSWER FOUR (4) QUESTIONS ONLY FROM FIVE (5) QUESTIONS.**  
**PART B: ANSWER ALL QUESTIONS.**

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THIS QUESTION PAPER CONSISTS OF ELEVEN (11) PAGES

**PART A**

**Q1** (a) What is “District Cooling Systems” (DCS) and explain objectives and advantages of DCS?

(8 marks)

(b) Explain and sketch a diagram for all water system and air-water system in an air conditioning system.

(12 marks)

**Q2** (a) ASHRAE Standard 55-2004, “Thermal Environmental Conditions for Human Occupancy,” is a revision of Standard 55-1992. The standard specifies conditions in which a specified fraction of the occupants will find the environment thermally acceptable. Thermal comfort is perceived as the comfort of human beings under given room conditions. Elevated airspeed is one of the factors for thermal comfort. Explain five (5) local thermal discomfort because of elevated speed.

(20 marks)

**Q3** (a) What are different between humidity ratio and relative humidity?

(4 marks)

(b) Using the psychrometric chart in **Figure Q3 (b)** to complete the properties as shown as in **Table Q3 (b)**.

(8 marks)

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(c) An air conditioning unit has a cooling coil that cools and dehumidifies 25,000 CFM of air from 84 °F DB and 60% RH to 65 °F DB and 55 °F DP. Find the flow rate in lb/hr, sensible, latent, and total capacity of the cooling coil (Plot the process on **Figure Q3 (c)** (Psychrometric chart).

Given, For air 1 CFM = 4.5 lb/hr.

(8 marks)

- Q4** (a) What is the different between heat lost and heat gain?  
(4 Marks)
- (b) Solar radiation through glass and lighting are heat gain components to contribute to the room cooling load. Finds cooling load for a building wall facing southwest has a window area of 300 ft<sup>2</sup>. The glass is ¼ in. single clear glass with-colored interior venetian blinds. The building is of medium construction and is located at 40°N latitude. Find the solar cooling load, if maximum “solar heat gain factor (SHGF)” is 196 BTU/hr-ft<sup>2</sup>, “Shading coefficient (SC)” is 0.67, and cooling load factor for glass is 0.83.  
(8 Marks)
- (c) A room has 130 ft<sup>2</sup> of single glass windows with vinyl frame. Inside air temperature is 70 °F and outdoor average temperature is on a design day is 90 °F. Find cooling load due to conduction heat gain through the windows at 2 PM Daylight saving time with cooling load temperature different 12 °F, correction for latitude and month is -1 and overall heat transfer coefficient (U) for glass is 0.9 BTU/HR-FT<sup>2</sup>-F.  
(8 Marks)
- Q5** (a) A fan is a gas flow producing machine which operates on the same basic principles as a centrifugal pump or compressor. A fan is similar to a centrifugal pump or compressor, which is convert mechanical rotated energy, applied at the shafts, to gas (or fluid) energy. Explain, the following terms for fan performance:
- (i) fan Volume
  - (ii) fan Outlet Velocity
  - (iii) fan Velocity Pressure
  - (iv) fan Total Pressure
- (4 marks)

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- (b) The duct shown in **Figure Q5 (b)** has 7000 CFM flowing through it. The friction loss from point 1 to 2 is 0.43 in.w. If the static pressure at 1 is 1.23 in w.g., determine the static pressure at point 2?

(6 marks)

- (c) The steel piping system as shown as in **Figure Q5 (c)** is a closed system. According to the energy equation, the required pump head rise from  $F$  to  $A$  is equal to the pressure drop due to friction loss from  $A$  to  $F$  through the system. But to find pump head only the circuit with the greatest pressure drop is chosen through the longest circuit,  $ABCC'DEF$ , Circuits  $ABEF$  and  $ACDF$  are ignored. Using **Table Q5 (c) i**, **Table Q5 (c) ii** and **Figure Q5 (c) ii**, determine the required pump head for the piping system.

(10 marks)

**PART B**

- Q6 (a)** According to ASHRAE 2009, HVAC equipment for a building is one of the major sources of interior noise, and its effect on the acoustical environment. Further, noise from equipment located outdoors often propagates to the community. Explain the glossary of acoustic terminology as below:

- i. Air-borne Noise
- ii. Frequency
- iii. Noise
- iv. Sound Power Level
- v. Sound Pressure Level


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(10 marks)

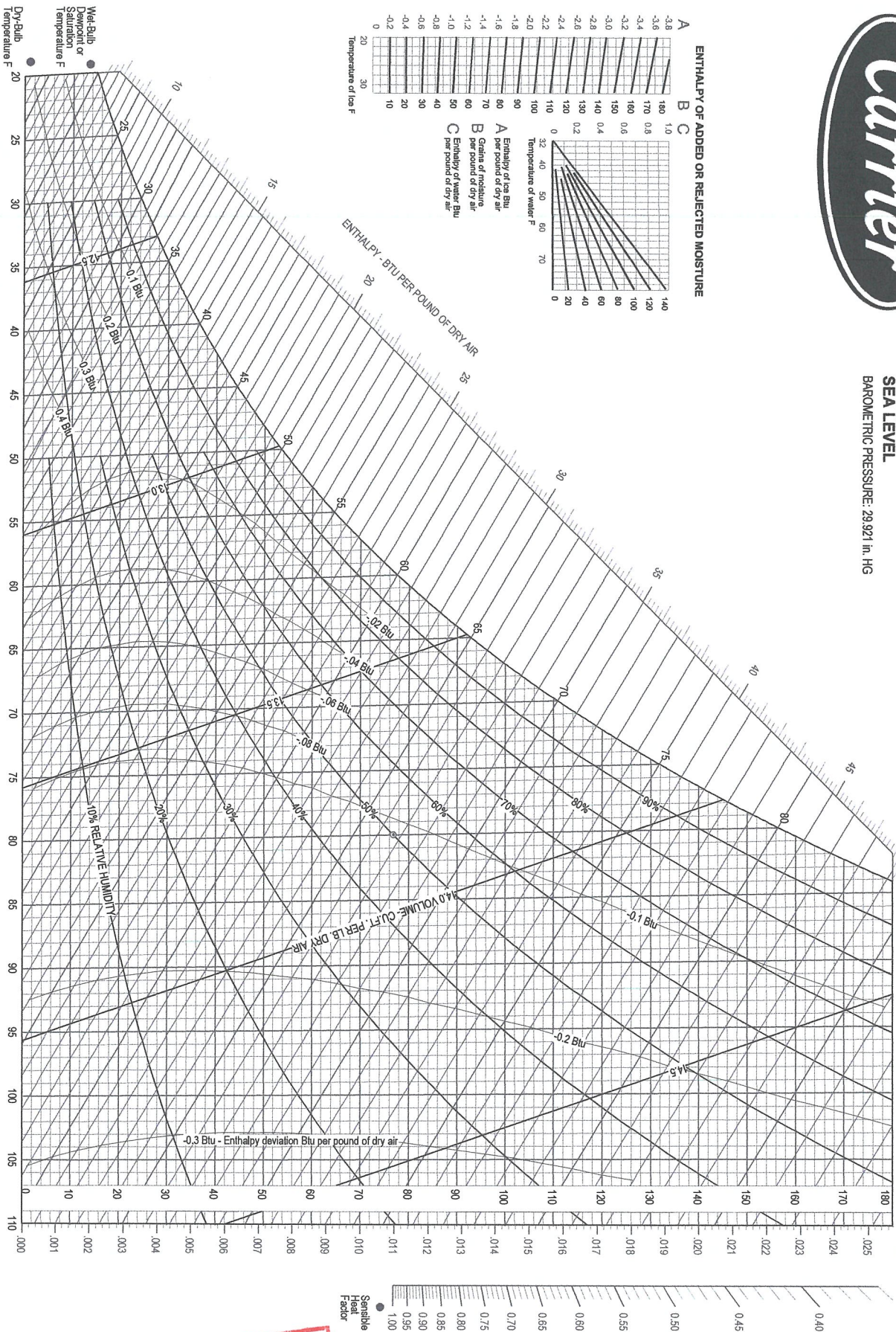
- (b) The noise problem associated with fans may cause noise disturbance to nearby residents (see **Figure Q6 (b)**). It mainly comes from the interaction of flow turbulence and solid surface of fan blades, and blade / fan vibration. The noise is transmitted upstream and downstream in the connecting ducts or to the atmosphere through the fan case. Give two practical remedies to solve this problem. Illustrate the systems with the appropriate figures.

(10 marks)

- END OF QUESTION -



PSYCHROMETRIC CHART Normal Temperature I-P Units SEA LEVEL BAROMETRIC PRESSURE: 29.921 in. HG



Grains of moisture per pound of dry air

Pounds of moisture per pound of dry air

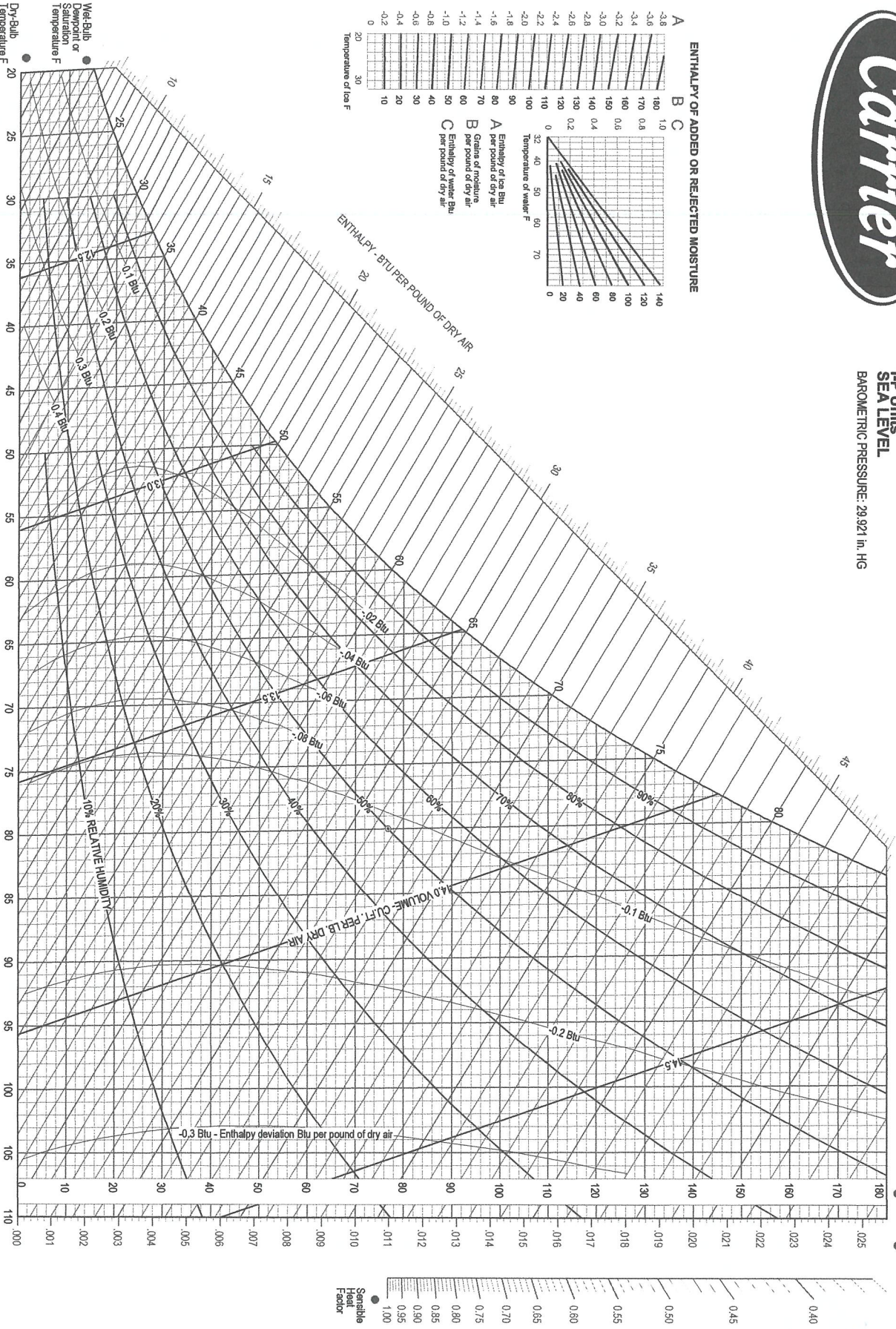
Sensible Heat Factor

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Figure Q3(b): Psychrometric chart



PSYCHROMETRIC CHART Normal Temperature I-P Units SEA LEVEL BAROMETRIC PRESSURE: 29.921 in. HG



Below 32 F, properties and enthalpy deviation lines are for ice.

Figure Q3(c): Psychrometric chart

Chart by: HANDS DOWN SOFTWARE, www.handsdownsoftware.com

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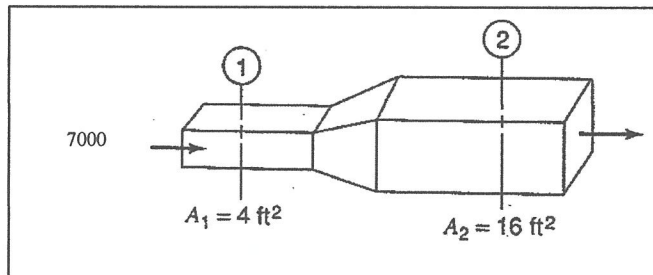
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**PART A**

**Table: Q 3 (b)**

| No. | DB,F | WB,F | DP,F | RH,% | W.gr/ lb<br>d.a | h, Btu/lb |
|-----|------|------|------|------|-----------------|-----------|
| i   | 80   | 60   |      |      |                 |           |
| ii  | 75   |      | 40   |      |                 |           |
| iii |      | 65   | 50   |      |                 |           |
| iv  |      |      | 50   | 40   |                 |           |



**Figure 5(b): Air Flow in the Ducting**

**Given:**

1 in w = 69.6 ft. air

Gravity,  $g = 32.2 \text{ ft. /sec}^2$

$$\frac{1 \text{ in. w}}{69.6 \text{ ft. air}}$$

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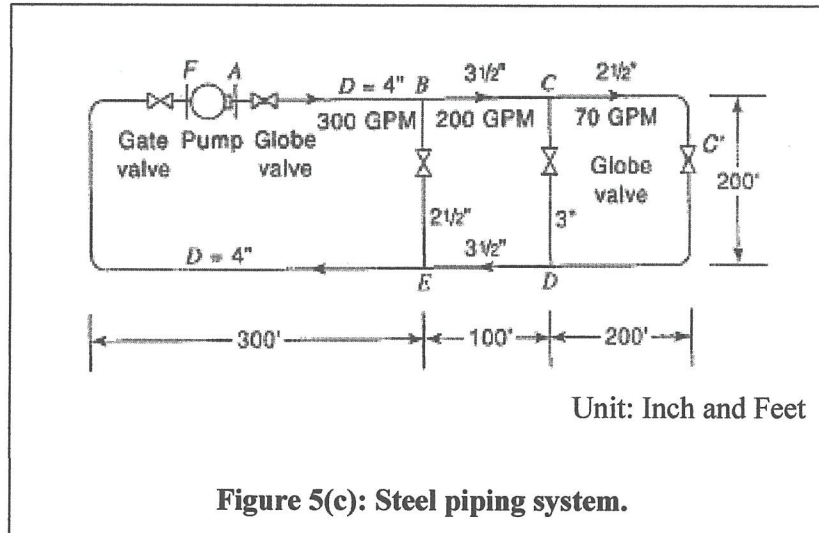
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**Table 5(c) i: Piping pressure drop calculation.**

| Section | Item | D,in | GPM | V<br>FPS | E.L.,<br>ft | No.of<br>items | Total<br>length,ft | Friction loss $H_f$ |                |
|---------|------|------|-----|----------|-------------|----------------|--------------------|---------------------|----------------|
|         |      |      |     |          |             |                |                    | Ft w./100ft         | Total<br>ft w. |
|         |      |      |     |          |             |                |                    |                     |                |
|         |      |      |     |          |             |                |                    |                     |                |
|         |      |      |     |          |             |                |                    |                     |                |
|         |      |      |     |          |             |                |                    |                     |                |
|         |      |      |     |          |             |                |                    |                     |                |

*Note:*

D: Diameter Pipe

FPS: Feet per second

in: Inch

E.L: Elevation

GPM: Gallon per minute

ft: feet

V: Velocity

Ft w: Feet water

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**Table 5(c) ii: Equivalent Feet of pipe for fittings and valves**

|                           | Nominal Pipe Size (inches) |     |     |     |     |     |     |     |      |      |      |      |      |
|---------------------------|----------------------------|-----|-----|-----|-----|-----|-----|-----|------|------|------|------|------|
|                           | ½                          | ¾   | 1   | 1¼  | 1½  | 2   | 2½  | 3   | 4    | 5    | 6    | 8    | 10   |
| 45° Elbow                 | 0.8                        | 0.9 | 1.3 | 1.7 | 2.2 | 2.8 | 3.3 | 4.0 | 5.5  | 6.6  | 8.0  | 11.0 | 13.2 |
| 90° Elbow standard        | 1.6                        | 2.0 | 2.6 | 3.3 | 4.3 | 5.5 | 6.5 | 8.0 | 11.0 | 13.0 | 16.0 | 22.0 | 26.0 |
| 90° Elbow long            | 1.0                        | 1.4 | 1.7 | 2.3 | 2.7 | 3.5 | 4.2 | 5.2 | 7.0  | 8.4  | 10.4 | 14.0 | 16.8 |
| Gate valve open           | 0.7                        | 0.9 | 1.0 | 1.5 | 1.8 | 2.3 | 2.8 | 3.2 | 4.5  | 6.0  | 7.0  | 9.0  | 12.0 |
| Globe valve open          | 17                         | 22  | 27  | 36  | 43  | 55  | 67  | 82  | 110  | 134  | 164  | 220  | 268  |
| Angle valve               | 7                          | 9   | 12  | 15  | 18  | 24  |     |     |      |      |      |      |      |
| Tee—side flow             | 3                          | 4   | 5   | 7   | 9   | 12  | 14  | 17  | 22   | 28   | 34   | 44   | 56   |
| Swing check valve         | 6                          | 8   | 10  | 14  | 16  | 20  | 25  | 30  | 40   | 50   | 60   | 80   | 100  |
| Tee—straight through flow | 1.6                        | 2.0 | 2.6 | 3.3 | 4.3 | 5.5 | 6.5 | 8.0 | 11.0 | 13.0 | 16.0 | 22.0 | 26.0 |
| Radiator angle valve      | 3                          | 6   | 8   | 10  | 13  |     |     |     |      |      |      |      |      |
| Diverting tee             |                            | 20  | 14  | 11  | 12  | 14  | 14  | 14  |      |      |      |      |      |
| Flow check valve          |                            | 27  | 42  | 60  | 63  | 83  | 104 | 125 | 126  |      |      |      |      |
| Air eliminator            |                            | 2   | 3   | 4   | 5   | 7   | 8   | 13  | 15   |      |      |      |      |
| Boiler (typical)          | 5                          | 7   | 9   | 11  | 13  | 17  |     |     |      |      |      |      |      |

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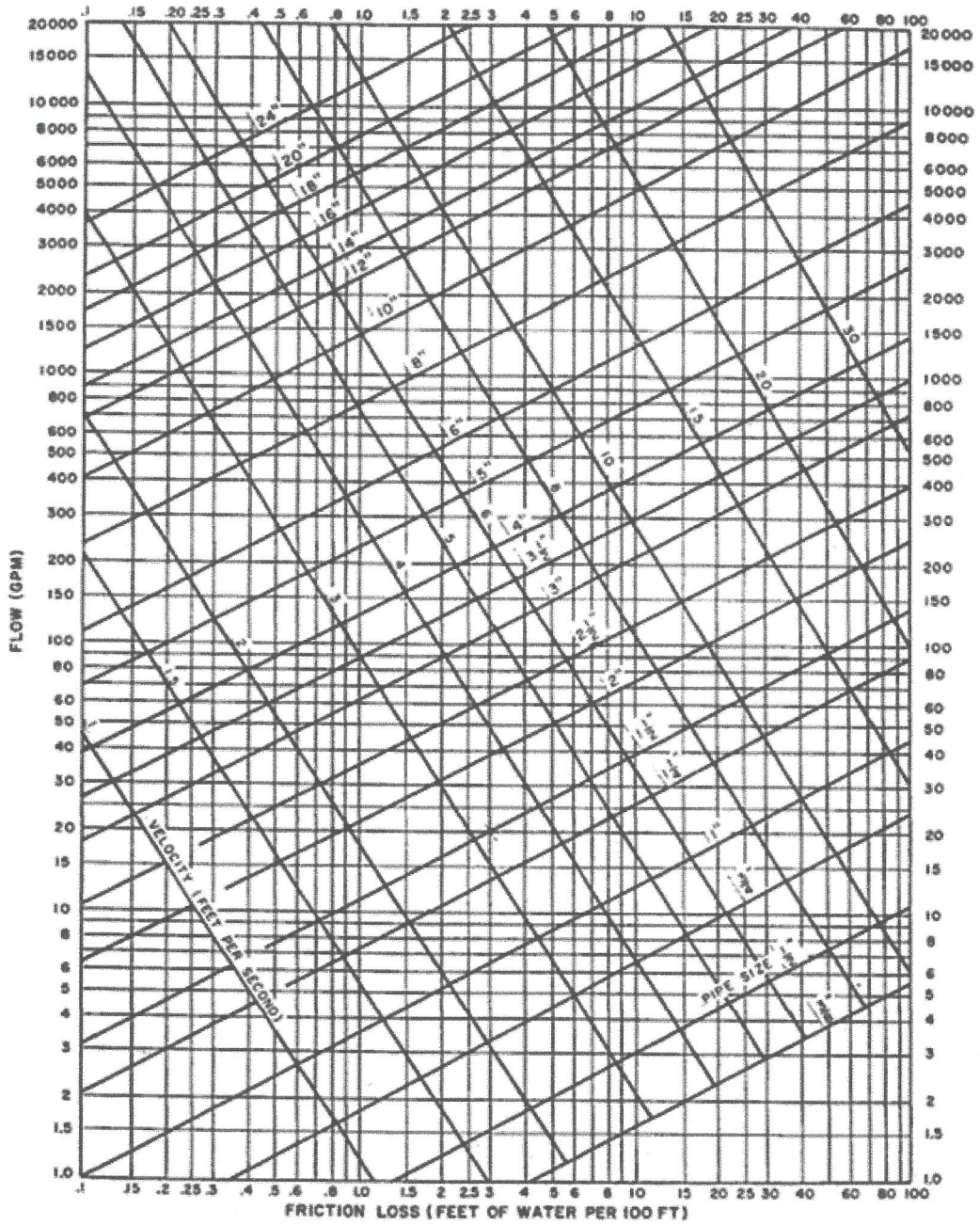


Figure 5 (c) ii: Friction loss for water in Schedule 40 Steel Pipe-Closed system (Courtesy: Carrier Corporation, Syracuse, NY)

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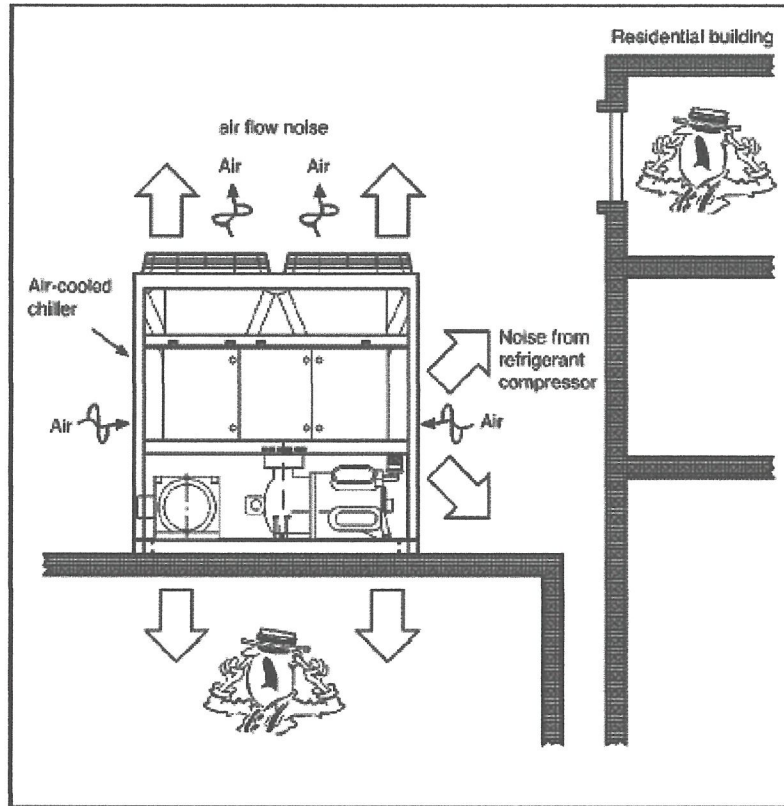


Figure 6 (b) ii: Air-borne Noise from Air-cooled Chillers

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