



UNIVERSITI TUN HUSSEIN ONN MALAYSIA

FINAL EXAMINATION SEMESTER I SESSION 2009/2010

SUBJECT	:	DYNAMICS
SUBJECT CODE	:	BDA 2013
COURSE	:	2 BDD
DATE	:	NOVEMBER 2009
DURATION	:	2 ½ HOURS
INSTRUCTION	:	ANSWER FOUR (4) OUT OF SIX (6) QUESTIONS

THIS PAPER CONSISTS OF 8 PAGES

Q1 The large window in **Figure Q1** is opened using a hydraulic cylinder AB. The cylinder extends at a constant rate of 0.5 m/s.

- Relate the coordinate θ and s using the law of cosines.
- Determine the time derivatives of equation obtained in (a).
- At the instant $\theta = 40^\circ$, find the value of s from the equation obtained in (a).
- Calculate the angular velocity, ω of the window at the instant $\theta = 40^\circ$.
- Determine the time derivatives of equation obtained in (c).
- Calculate the angular acceleration, α of the window at the instant $\theta = 40^\circ$

[25 marks]

Q2 The attached wheels roll without slipping on the plates A and B , which are moving in opposite directions as shown in **FIGURE Q2**. If $v_A = 60 \text{ mm/s}$ to the right and $v_B = 200 \text{ mm/s}$ to the left, determine;

- the velocity of the center O .
- the velocity of the point P .

[25 marks]

Q3 **FIGURE Q3** shows the slider A oscillates in the slot about the neutral position O with frequency of 2 cycles per second and an amplitude x_{max} of 50 mm so that its displacement in millimeters may be written as $x = 50 \sin 4\pi t$ where t is the time in seconds. The disc, in turn, is set into angular oscillation about O with frequency of 4 cycles per second and an amplitude $\theta_{max} = 0.2 \text{ rad}$. The angular displacement is thus given by $\theta = 0.2 \sin 8\pi t$. Calculate the acceleration of A for the positions

- $x = 0$ with \dot{x} positive.
- $x = 50 \text{ mm}$.

[25 marks]

Q4 **FIGURE Q4** shows the pendulum which is suspended at point O and consists of two slender rod AB and OC. The slender rods have the mass of 4 kg/m . The disk is fixed at the ends of slender rod OC and it has a density of 6000 kg/m^3 and a thickness of 15 mm . The disk has the radius of 300 mm while the hole has the radius of 150 mm . Determine;

- (a) The moment inertia of rod OC and BC passing through the end point O.
- (b) The moment inertia of the disk about the end point O.
- (c) The pendulum moment of inertia about an axis passing through the pin at O.
- (d) The location of y of the center of mass G of the pendulum.
- (e) The pendulum moment of inertia about an axis passing through the mass center G of the pendulum.

[25 marks]

Q5 A plate shown in **FIGURE Q5** is suspended at A and B. Calculate,

- (a) Mass of the plate.
- (b) Location of the center of gravity of the plate.
- (c) Mass moment of inertia of the plate about the center of gravity.
- (d) Mass moment of inertia of the plate about the point of rotation at A.

When the string at B is immediately cut, determine,

- (e) The angular acceleration of the plate.
- (f) The horizontal and vertical components of the reaction at pin A.

[25 marks]

Q6 The double pulley shown in **FIGURE Q6** has a mass of 15 kg and a centroidal radius of gyration of 165 mm . Cylinder A and B are attached to the cords that are wrapped on the pulleys. The coefficient of kinetic friction between block B and the surface is 0.25 . Knowing that system is released from the position illustrated in **FIGURE Q6**, determine;

- (a) The velocity of the cylinder A as it strikes the ground
- (b) The total distance block B moves before coming to rest.
(*Hints: think about energy method to solve this problem*)

[25 marks]

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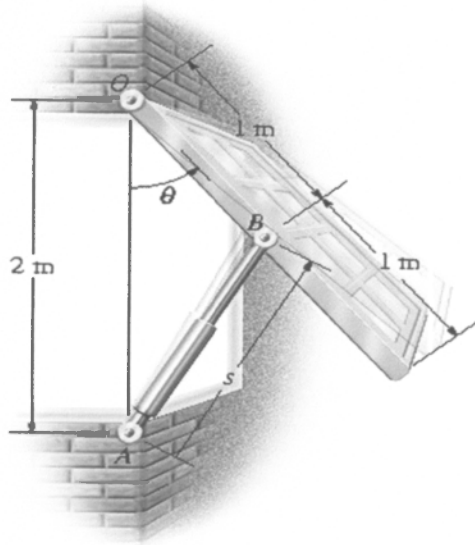


FIGURE O1 – Hydraulic Window

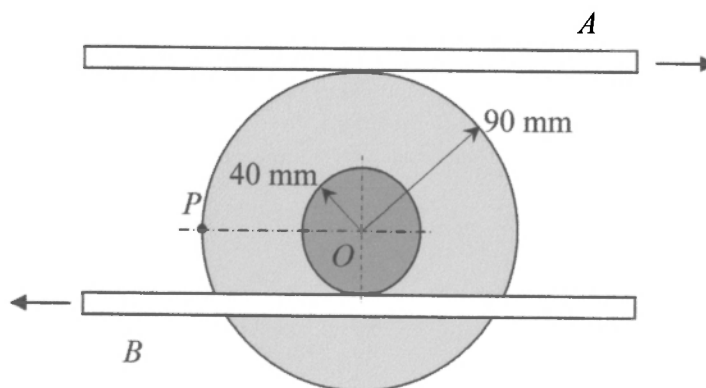


FIGURE O2 – Rolling Wheel on the Plates

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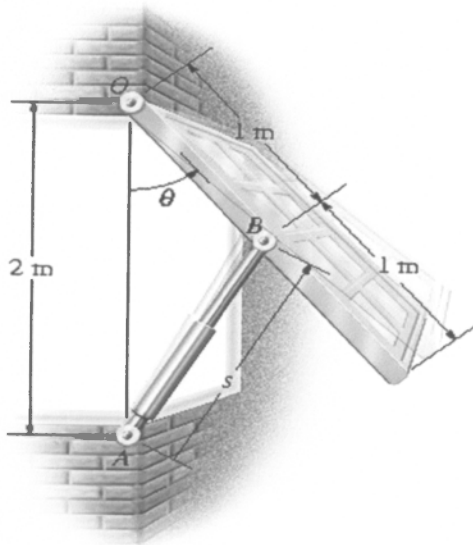


FIGURE O1 – Hydraulic Window

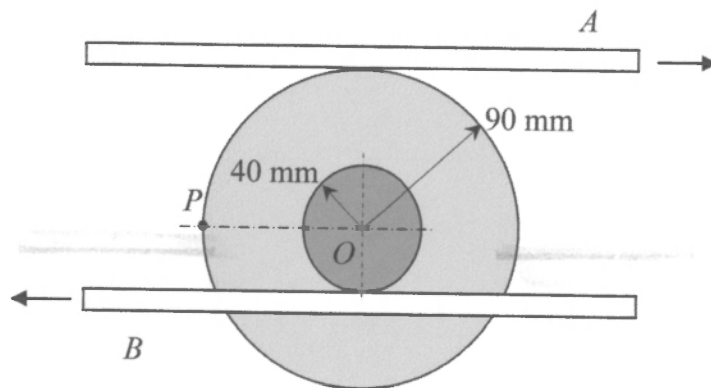


FIGURE O2 – Rolling Wheel on the Plates

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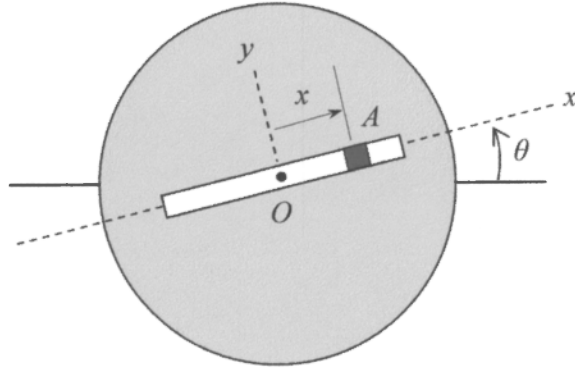


FIGURE O3 – Slider ‘A’ and the Disc.

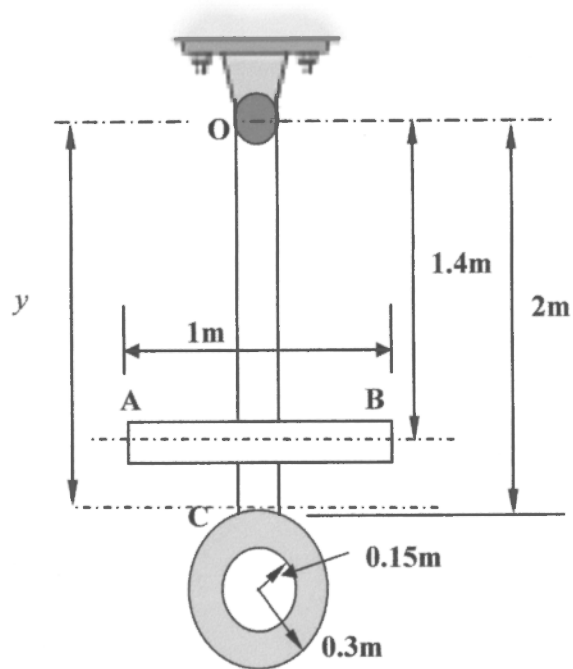


FIGURE O4 – A Pendulum

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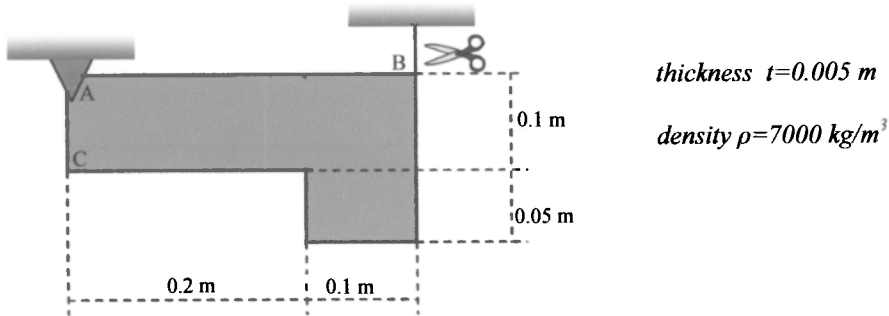


FIGURE Q5 – Hanging Plate

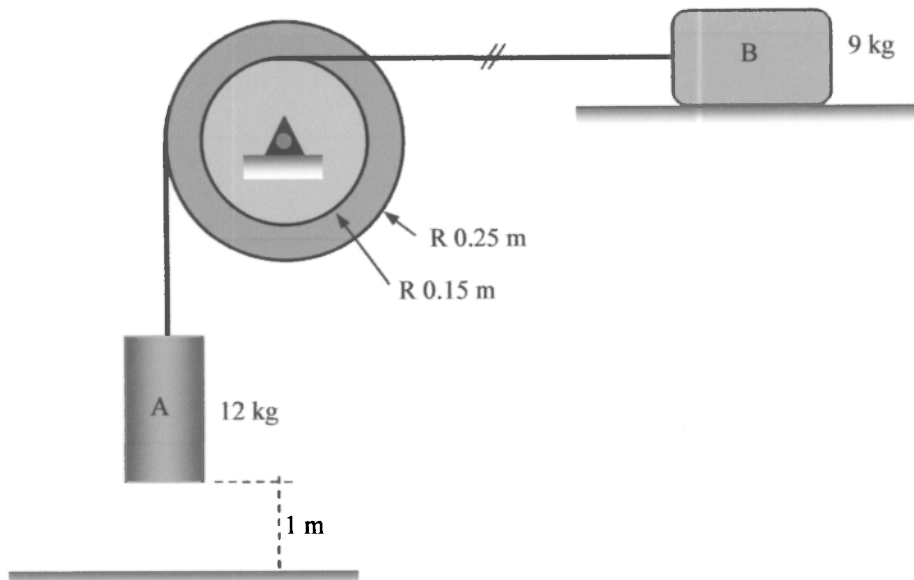


FIGURE Q6 – Double Pulley

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$$s = s_0 + v_0 t + \frac{1}{2} a t^2$$

$$v = v_0 + a t$$

$$v^2 = v_0^2 + 2 a s$$

$$\theta = \theta_0 + \omega t + \frac{1}{2} \alpha t^2$$

$$\omega = \omega_0 + \alpha t$$

$$\omega^2 = \omega_0^2 + 2 \alpha s$$

$$\mathbf{v} = \mathbf{v}^r + \mathbf{v}^\theta$$

$$\mathbf{v}^\theta = r \omega \quad \mathbf{v}^r = \dot{r}$$

$$\mathbf{a} = \mathbf{a}^r + \mathbf{a}^\theta$$

$$\mathbf{a}^r = \ddot{r} - \dot{\theta}^2 r$$

$$\mathbf{a}^\theta = \ddot{\theta} r + 2 \dot{\theta} \dot{r}$$

$$\mathbf{a} = \mathbf{a}^n + \mathbf{a}^t$$

$$\mathbf{a}^n = r \omega^2 = \frac{v^2}{r}$$

$$\mathbf{a}^t = r \alpha$$

$$T_1 + U_{1 \rightarrow 2} = T_2$$

$$T_1 + V_1 = T_2 + V_2$$

$$U = \Delta T + \Delta V_g + \Delta V_e$$

$$\Delta T = \frac{1}{2} m (v_2^2 - v_1^2) + \frac{1}{2} I_G (\omega_2^2 - \omega_1^2)$$

$$\Delta V_g = m g (h_2 - h_1)$$

$$\Delta V_e = \frac{1}{2} k (x_2^2 - x_1^2)$$

$$m v_1 + \sum_{t_1}^{t_2} \int F dt = m v_2$$

$$(H_0)_1 + \sum_{t_1}^{t_2} \int M_0 dt = (H_0)_2$$

$$m_A (v_A)_1 + m_B (v_B)_1 = m_A (v_A)_2 + m_B (v_B)_2$$

$$I_G \omega_1 + m (v_G)_1 d_1 + \sum \int M_A dt = I_G \omega_2 + m (v_G)_2 d_2$$

$$e = - \frac{(v_B)_2^n - (v_A)_2^n}{(v_B)_1^n - (v_A)_1^n}$$

$$(v_A)_1^t = (v_A)_2^t$$

$$\sum M_G = I_G \alpha$$

$$\sum F = m a$$

$$\mathbf{v}_P = \mathbf{v}_{P'} + \mathbf{v}_{P/Oxy}$$

$$\mathbf{v}_P = (\ddot{\mathbf{r}})_{OXY} = \boldsymbol{\Omega} \times \mathbf{r} + (\ddot{\mathbf{r}})_{Oxy}$$

$$\mathbf{a}_P = \mathbf{a}_{P'} + \mathbf{a}_{P/Oxy} + \mathbf{a}_C$$

$$\mathbf{a}_P = \boldsymbol{\Omega} \times (\boldsymbol{\Omega} \times \mathbf{r}) + \dot{\boldsymbol{\Omega}} \times \mathbf{r} + 2(\boldsymbol{\Omega} \times \dot{\mathbf{r}})_{Oxy} + (\ddot{\mathbf{r}})_{Oxy}$$

$$I = m k_G^2$$

$$I = \int_m r^2 dm$$

$$I_{XX} = I_{YY} = \frac{1}{4} m r^2$$

$$I_{ZZ} = \frac{1}{2} m r^2$$

$$I_{XX} = I_{YY} = \frac{1}{12} m l^2$$

$$I_{X'X'} = \frac{1}{3} m l^2$$

$$I_{XX} = \frac{1}{12} m (B^2 + C^2)$$

$$I_{YY} = \frac{1}{12} m (A^2 + B^2)$$

$$I_{ZZ} = \frac{1}{12} m (A^2 + C^2)$$

