

UNIVERSITI TUN HUSSEIN ONN MALAYSIA

FINAL EXAMINATION SEMESTER I SESSION 2023/2024

COURSE NAME

: MECHANICS OF MACHINES

COURSE CODE

: DAM 23803

PROGRAMME CODE

: DAM

EXAMINATION DATE

: JANUARY / FEBRUARY 2024

DURATION

: 3 HOURS

INSTRUCTION

1. ANSWER FIVE (5) QUESTIONS ONLY

2. THIS FINAL EXAMINATION IS CONDUCTED VIA CLOSED BOOK

3. STUDENTS ARE **PROHIBITED** TO CONSULT THEIR OWN MATERIAL OR ANY EXTERNAL RESOURCES DURING THE EXAMINATION CONDUCTED VIA CLOSED BOOK

THIS QUESTION PAPER CONSISTS OF TEN (10) PAGES

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Q1 (a) In gear terminology, circular pitch, diametral pitch and module of the gear are important parameters. Explain how those three (3) parameters related to each other.

(6 Marks)

(b) State the four types of gears and their meaning.

(4 Marks)

(c) A motor is accelerating a 230 kg load with an acceleration of 1.0 m/s² through a gear system as shown in **Figure Q1(c)**. The rope that carries the load are encircled on a hoist with a diameter 1.2 m. Gear for the hoist's shaft has 150 teeth, gear for motor shaft has 20 teeth. Gear efficiency is 85%. Mass of motor shaft is 230 kg with a radius of gyration of 110 mm while hoist shaft has 1000 kg of mass with radius of gyration of 500 mm. Calculate the torque of the motor needed to bring up the load with acceleration 1.0 m/s². Neglect friction effect.

(10 Marks)

Q2 (a) Explain three (3) advantages of flat belt and three (3) advantages of V-belt.

(6 Marks)

- (b) In power transmission, belt-drive system depends on friction. However, slippage of the belt contributes to the inefficiency of the drive.
 - (i) Describe the phenomena of belt slip.

(2 Marks)

(ii) Belt slip condition considered as one of the factors that contribute to the inefficiency of the drive. Justify the consequences of that condition.

(2 Marks)

- (c) As shown in **Figure Q2(c)**, a V-belt drive with a groove angle of 40° is required to transmit 25 kW from a pulley 150 mm in diameter to another pulley 280 mm in diameter rotating at 200 rev/min. The coefficient of friction between the belt and the pulley is 0.3 and the maximum permissible belt tension is 1200 N for each of the ten belts used.
 - (i) Calculate the torque on the smaller pulley.

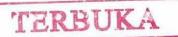
(3 Marks)

(ii) Determine the angle of contact.

(4 Marks)

(iii) Suggest the ideal center distance.

(3 Marks)



Q3 (a) Explain the static and dynamic balancing and provide two (2) examples of each.

(6 Marks)

(b) State the principle of D'Alembert.

(2 Marks)

(c) Figure Q3(c) shows four masses A, B, C and D are to be completely balanced at radius 120 mm, 150 mm and 180 mm and 210 mm respectively. Mass B, C and D are 20 kg, 15 kg and 5 kg respectively. The planes which the mass revolve are spaced 100 cm apart. Find the required mass A and the angular settings relative to a plane so that the shaft is in complete balance.

(12 Marks)

(a) Describe the differences between static friction and dynamic friction with examples.

(6 Marks)

(b) The mean diameter of a 'Vee' threaded screw jack as shown in Figure Q4(b) is 70 mm. The pitch of the thread is 20 mm and the angle of thread β is 22°. The coefficient of friction is 0.17. Calculate the efficiency and force that must be applied at the end of a 0.75 m long lever, which is perpendicular to the longitudinal axis of screw to raise a load of 40 kN.

(14 Marks)

(a) Indicate the velocity of pin joint as illustrated in Figure Q5(a). The radius of the pin joint is given as r.

(4 Marks)

- (b) ABCD is a four-bar chain with the link AD is fixed as Figure Q5(b). The length of the links are AB = 6.25 cm, BC = 17.5 cm, CD = 11.25 cm, DA = 20 cm. The crank AB makes 100 rpm in the clockwise direction. Find the following when the angle BAD is 60°.
 - (i) The angular velocity of links CD and BC.

(12 Marks)

(ii) Velocity of point E, 10cm from C on the link BC.

(2 Marks)

(iii) The velocity of point F, which is 10.5 cm from B and C and lying outside ABCD.

(2 Marks)

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Q6 (a) Define natural frequency for undamped free vibration and list two (2) ways for determining it.

(4 Marks)

- (b) A uniform thin rod AB as shown in **Figure Q6(b)** is hinged at point A. The moment of inertia of the thin rod at point A is 80 kgm². A concentrated mass of 3 kg is then fixed at point C and balanced horizontally by two springs positioned at point D and E. Given the spring stiffness of spring D and E are 3.5 kN and 1.5 kN respectively.
 - (i) Determine the moment of inertia of the complete assembly at point A,

(2 Marks)

(ii) Determine the force experienced by both spring in term of θ where θ is the angle of deflection.

(6 Marks)

(iii) Calculate the natural frequency of the complete assembly

(8 Marks)

-END OF QUESTIONS -





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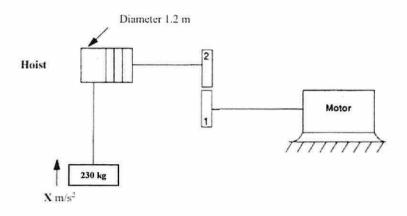


Figure Q1(c)

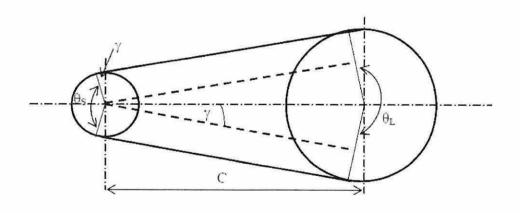


Figure Q2(c)

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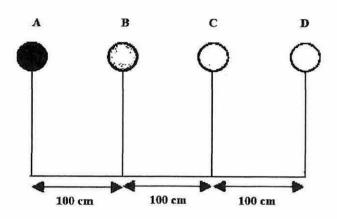


Figure Q3(c)

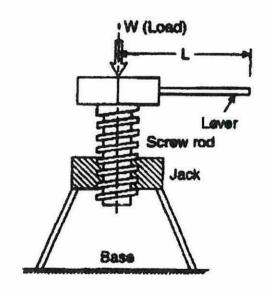


Figure Q4(b)

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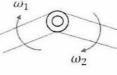
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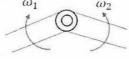
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(a) ω_1

(c)



(d)

Figure Q5(a)

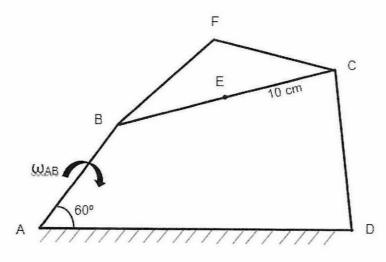


Figure Q5(b)

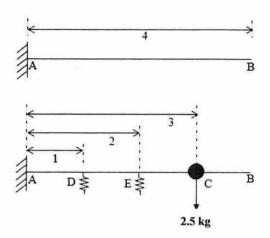
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Initial condition

Final condition

Figure Q6(b)



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- 1. Linear velocity at the contact surface of gear, $\pi D_1 N_1 = \pi D_2 N_2$
- $I_{equiv} = \left(I_A + \frac{I_B n^2}{\eta_G}\right)$ 2. Equivalent Moment of Inertia,
- $n = \frac{N_2}{N_1} = \frac{d_1}{d_2}$ 3. Velocity Ratio for belt drives,
- $\frac{T_{_{1}}}{T_{,}}=e^{u\theta}$ 4. Belt tension ratio for flat belt,
- $\frac{T_1}{T} = e^{\left(\frac{\mu\theta}{\sin\theta}\right)} = e^{(\mu\theta)(\cos\epsilon\epsilon\theta)}$ 5. Belt tension ratio for V-Belt,
- 6. V-Belt type force balance, $R_{N} = \frac{R}{2 \sin \beta}$
- 7. Maximum Power for Belt Drives, $P = (T_1 T_2)v$
- 8. Centrifugal force term, $\rho A v^2 = T_c$
- 9. Limiting Angle of Friction, $\tan \phi = \frac{F}{R} = \mu$
- 10. Inclination of Square Threaded Screw, $\tan \alpha = \frac{p}{\pi d}$
- 11. Efficiency for Square Threaded Screw, $\eta = \frac{p}{\pi D \tan(\beta + \alpha)}$
- 12. Radial component of acceleration, $f_{BA}^{r} = \omega^{2}(BA) = \frac{(V_{BA})^{2}}{DA}$

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13. Tangential component of acceleration, $f_{BA}^{t} = \alpha(BA)$

14. Newton's Second Law of Motion, $\sum M_O = I_O \ddot{\theta}$

15. Principle of conversion of energy, $\frac{d}{dt}[T.K + T.U]$