

## UNIVERSITI TUN HUSSEIN ONN MALAYSIA

## FINAL EXAMINATION SEMESTER I SESSION 2022/2023

:

**COURSE NAME** 

AUTOMOTIVE ENGINE

**TECHNOLOGY** 

**COURSE CODE** 

BNG 30403

PROGRAMME CODE :

**BNG** 

EXAMINATION DATE :

FEBRUARY 2023

DURATION

3 HOURS

INSTRUCTION

1. ANSWER ALL QUESTIONS

2. THIS FINAL EXAMINATION IS CONDUCTED VIA **CLOSED** 

BOOK.

3. STUDENTS ARE **PROHIBITED**TO CONSULT THEIR OWN
MATERIAL OR ANY EXTERNAL
RESOURCES DURING THE
EXAMINATION CONDUCTED VIA

CLOSED BOOK.

THIS OUESTION PAPER CONSISTS OF FIVE (5) PAGES

CONFIDENTIAL



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- Q1 (a) Explain and compare Spark Ignition and Compression Ignition engines with respect to
  - (i) fuel used;
  - (ii) ignition process;
  - (iii) compression ratio;
  - (iv) efficiency; and
  - (v) weight.

(5 marks)

(b) Explain the reason for the operation of forcing additional pressurized air in the engine cylinder with supercharging equipment

(5 marks)

- (c) A 4 cylinder, 2 stroke diesel engine with 11 .5 cm bore and 13.8 cm stroke produces 98 kW of brake power at 2600 rpm. The compression ratio, r<sub>c</sub> is 17:1. Calculate:
  - (i) the engine displacement (cm<sup>3</sup>);
  - (ii) brake mean effective pressure (kPa);
  - (iii) engine torque (Nm); and
  - (iv) clearance volume of one cylinder (cm<sup>3</sup>).

(15 marks)

Q2 (a) In engines combustion, explain the influence of intake air temperature on engine efficiency.

(3 marks)

(b) Explain the Premixed Charge Compression Ignition (PCCI) combustion as shown in **FIGURE Q2** (b) (introduced by Toyota) and how it is improve the combustion process in order to reduce the exhaust emissions.

(5 marks)

- (c) A two liter engine of six-cylinder that operates on four stroke cycle at 3800 RPM (revolutions per minute). The compression ratio, r<sub>c</sub> is 12, the length of the connecting rods, r is 15.5 cm, and the engine is square (B=S), B = bore, S = stroke. At this speed, combustion ends at 20° TDC (Top-Dead-Center). Calculate:
  - (i) cylinder bore, B and stroke length, S;
  - (ii) average piston speed  $\overline{U}_{p}$ ;
  - (iii) clearance volume of one cylinder  $V_c$ ;
  - (iv) crank offset R;
  - (v) piston speed at the end of combustion  $U_p$ ;
  - (vi) distance the piston has traveled from TDC at the end of combustion, x; and
  - (vii) volume in the combustion chamber at the end of combustion V.

(17 marks)



Q3 (a) Explain the influences of the fuel type in engine design and state **THREE** (3) basic engine designs together with their configuration.

(3 marks)

- (b) Define the following matters with the aid of sketch:
  - (i) combustion chamber;
  - (ii) displacement volume;
  - (iii) cubic capacity;
  - (iv) clearance volume; and
  - (v) compression ratio.

(5 marks)

- (d) A four-cylinder, four-stroke, 1.6 L gasoline engine operates on the Otto cycle with a compression ratio of 10. The air is at 100 kPa and 37°C at the beginning of the compression process, and the maximum pressure in the cycle is 7.5 MPa. The compression and expansion processes may be modeled as polytropic with a polytropic constant of 1.3. Using constant specific heats at 850 K. the air-fuel ratio, defined as the amount of air divided by the amount of fuel intake, is 16. Determine:
  - (i) the temperature at the end of the expansion processes;
  - (ii) the net work output and the thermal efficiency;
  - (iii) the mean effective pressure;
  - (iv) the engine speed for a net power output of 50 kW; and
  - (v) the specific fuel consumption (g/kWh).

(17 marks)

Q4 (a) Figure Q4(a) shows the influences of turbocharger that increasing the air density in combustion chamber. As seen in Figure 1 (a), it clearly illustrate the histories of combustion pressure p<sub>f</sub> and heat release rate dQ/dt together with nozzle needle lift NL against time, t from start of injection. Explain the influences of the changes of ambient density on combustion process.

(5 marks)

(b) Explain the operation of catalytic converters and how are they helpful in reducing HC, Carbon Monoxide (CO) and NOx emissions. Construct the catalytic converters configuration.

(5 marks)

- (c) A 4.5 liter, 4-cylinder, two-stroke S.I engine is connected to an electrical power generator of 250 volt. At 1800 rpm, the generator produces 55 amps of current with operating efficiency of 86%. Determine:
  - (i) The power output of the engine (in kW and hp); and
  - (ii) The engine torque (Nm).

(15 marks)

- END OF QUESTIONS -



3

## FINAL EXAM / PEPERIKSAAN AKHIR

SEMESTER / SESSION:

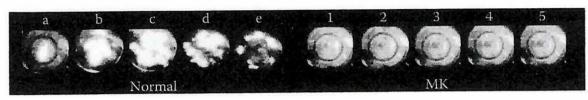
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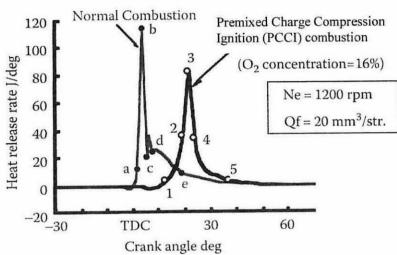
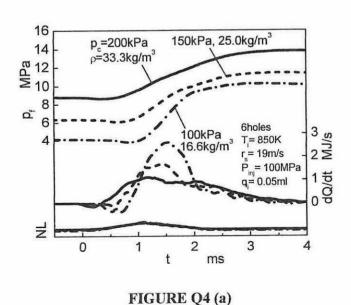


FIGURE Q2 (b)



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## FINAL EXAMINATION

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Power output motor (watt) = Power output engine (watt) = volts x amps

Piston speed,  $\overline{U_P} = 2SN$ 

ambient density (air),  $\rho_a = 1.181 kg/m^3$ 

Compression ratio,  $r_c$  is defined as:  $r_c = \frac{V_{BDC}}{V_{TDC}}$ ,  $r_c = \frac{(V_d + V_c)}{V_c}$ 

Instantaneous piston speed;  $U_p / U_p = (\pi/2) \sin \theta \left[ 1 + \left( \cos \theta / \sqrt{R^2 - \sin^2 \theta} \right) \right], R = r/a, a = S/2$ 

Piston position or the distance between the crank axis and wrist pin axis or piston is given by, s:

$$s = a\cos\theta + \sqrt{r^2 - a^2\sin^2\theta}$$

Where a = crankshaft offset, r = connecting rod length and  $\theta$  = crank angle, measure from the centerline and it is zero when the piston is at TDC

Distance from TDC, x = r + a - s

Instantaneous volume, V at any crank angle,  $\theta$ :  $\frac{V}{V_c} = 1 + \frac{1}{2} (r_c - 1) \left[ R + 1 - \cos \theta - \sqrt{R^2 - \sin^2 \theta} \right]$  Vc = clearance volume, R = r/a

For an engine with  $N_c$  cylinders, displacement volume,  $N_d$ :

$$V_d = V_{BDC} - V_{TDC}$$
  $V_d = N_c \left(\frac{\pi}{4}\right) B^2 S$  Where B = cylinder bore, S = stroke, S=2a

The cylinder volume at any crank angle is given by:  $V = V_c + \left(\frac{\pi B^2}{4}\right)(r + a - s)$ , Where  $V_c =$  clearance volume

Brake work of one revolution,  $W_b$ :  $W_b = 2\pi T$ ;  $W_b = \frac{V_d(bmep)}{n}$ , Where T = engine torque, bmep = brake mean effective pressure, n = number of revolutions percycle

$$mep = \frac{Wn}{V_d N}$$

Mean effective pressure;

Engine torque, T, for 2-stroke and 4-stroke cycles:  $T_{2-stroke} = \frac{V_d(bmep)}{2\pi} \qquad T_{4-stroke} = \frac{V_d(bmep)}{4\pi}$ 

Engine power,  $W = \frac{WN}{n}$ ,  $W = 2\pi NT$ , N =engine speed

Specific fuel consumption  $sfc = \frac{m_f}{W}$