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Universiti Tun Hussein Onn Malaysia

**UNIVERSITI TUN HUSSEIN ONN MALAYSIA**

**FINAL EXAMINATION**

**SEMESTER I  
SESSION 2021/2022**

COURSE NAME : AIR CONDITIONING SYSTEM DESIGN  
COURSE CODE : BBA40603  
PROGRAMME CODE : BBG  
EXAMINATION DATE : JANUARY / FEBRUARY 2022  
DURATION : 3 HOURS  
INSTRUCTION : 1. ANSWER ALL QUESTIONS  
2. THIS FINAL EXAMINATION IS A **ONLINE ASSESSMENT AND CONDUCTED VIA CLOSE BOOK.**

THIS QUESTION PAPER CONSISTS OF **TWENTY-ONE (21) PAGES**

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TERBUKA

**SECTION A (20 marks)**

- Q1.** Why is it necessary to calculate the cooling load of a room?
- A. It defines the shape and volume of the room.
  - B. It is unnecessary, as rooms are only calculated in terms of area
  - C. To estimate the capacity that is required for cooling the room.
  - D. None above
- Q2.** What is the Cooling Load?
- A. It is the amount of heat that must be added or be removed from a place so that we able to control its pressure
  - B. It is the amount of heat that must be added or be removed from a place so that we able to control its temperature
  - C. It is the amount of heat that must be added or be removed from a place so that we able to control its humidity
  - D. It is the amount of heat that must be added or be removed from a place so that we able to control its pressure & temperature
- Q3.** Select the main types of heat loads included in space load calculation.
- I. Transmission loads
  - II. Solar radiation
  - III. People
  - IV. Infiltration
  - V. Ventilation
- A. I, II, III, & IV
  - B. I, II, III, & V
  - C. I, III, IV, & V
  - D. II, III, IV, & V
- Q4.** What is transmission load?
- A. Transmission load is the total heat load calculation through the roof, and we also need to consider the effect of solar radiation.
  - B. Transmission load is the total heat load calculation through the walls, and we also need to consider the effect of solar radiation.
  - C. Transmission load is the total heat load calculation through the walls, roof, and floor, and we also need to consider the effect of solar radiation.
  - D. Transmission load is the total heat load calculation through the floor, and we also need to consider the effect of solar radiation.

- Q5.** When specifying how many tons of cooling capacity for a building's chiller, you should look at what cooling load?
- A. Overall building sensible cooling load
  - B. Space sensible plus latent cooling load
  - C. Overall building sensible plus latent cooling load
  - D. Space sensible cooling load only
- Q6.** As shown below, increasing the number of people in space, impacts the cooling load estimate with...
- A. no effect
  - B. increase the inside design temperature
  - C. increasing the cooling load
  - D. decreasing the cooling load
- Q7.** Assume you calculated the sensible and latent cooling loads for a dance studio and discovered that the sensible cooling load is 64,000 BTU/hr and the latent cooling load is 12,500 BTU/hr. What is the sensible heat ratio?
- A. 0.16
  - B. 0.55
  - C. 0.84
  - D. 1.20
- Q8.** To determine the design CFM for an air handler serving several zones using a variable air volume (VAV) system...
- A. one needs to determine the time that the maximum cooling load occurs for all the zones that the air handler serves and then use that value to calculate the CFM required.
  - B. It is done the same way as selecting the design cooling load for a constant air volume (CAV) system handler.
  - C. One needs to determine the max cooling load for each zone served by the air handler, add these values to get a total cooling load, and then use the total cooling load to calculate the CFM.
  - D. None of the above answers

**Q9.** What is the U-A value for a glass window unit (see **Table Q9**) with the condition as below?

*Size 5ft × 7ft vertical window*

*Winter condition*

*Insulating glass unit with a 1/4 in glass and 1/2 in. air space.*

*Interior shading*

**Table Q9: Overall Coefficients of Heat Transmission (U-Factor) of Windows and Skylights, Btu/(hr.ft<sup>2</sup>.F)**

Description	Exterior Vertical Panels				Exterior Horizontal Panels (Skylights)	
	Summer		Winter		Summer	Winter
	No Indoor Shade	Indoor Shade	No Indoor Shade	Indoor Shade		
Flat glass						
Single Glass	1.04	0.81	1.1	0.83	0.83	1.23
Insulating Glass, Double						
3/16 in. air space (0.125 in. glass)	0.65	0.58	0.62	0.52	0.57	0.70
1/4 in. air space (0.125 in. glass)	0.61	0.55	0.58	0.48	0.54	0.65
1/2 in. air space (0.25 in. glass)	0.56	0.52	0.49	0.42	0.49	0.59
1/2 in. air space						
low emittance coating						
e = 0.20	0.38	0.37	0.32	0.30	0.36	0.48
e = 0.40	0.45	0.44	0.38	0.35	0.42	0.52
e = 0.60	0.51	0.48	0.43	0.38	0.46	0.56
Insulating Glass, Triple						
1/4 in. air space	0.44	0.40	0.39	0.31		
1/2 in. air space	0.39	0.36	0.31	0.26		
Storm Windows						
1 in. to 4 in. air spaces	0.50	0.48	0.50	0.42		
Plastic Bubbles						
Single					0.80	1.15
Double					0.46	0.70

- A. 0.56
- B. 14.70
- C. 17.92
- D. 19.52



**Q10.** What is the CLTD value for a wall (see **Table Q10**) with the condition as below?

*Wall type A*

*Wall surface: Light-colored exterior surface south façade*

*Time & Month: 1600 in July*

**Table Q10: CLTDs for Sunlit Walls**

Group A Walls																				
Solar Time, hr																				
	1	2	3	4	5	6	7	8	9	10	11	12	13	14	15	16	17	18	19	20
<b>N</b>	9	8	7	6	5	4	3	2	2	2	3	4	4	6	7	8	9	11	12	12
<b>NE</b>	10	8	7	6	5	4	3	3	3	6	9	11	13	14	14	15	15	16	16	15
<b>E</b>	11	9	8	7	6	4	3	3	4	7	11	14	18	20	21	21	21	20	19	18
<b>SE</b>	11	9	8	7	6	4	3	3	3	5	7	11	14	17	19	20	21	20	19	19
<b>S</b>	12	10	8	7	6	4	3	3	2	2	2	3	6	8	11	14	16	18	19	19
<b>SW</b>	17	14	12	10	8	7	5	4	3	3	3	4	6	8	11	14	18	19	19	19
<b>W</b>	19	17	14	12	9	8	6	4	4	3	3	4	6	7	9	12	17	21	24	24
<b>NW</b>	16	14	12	9	8	6	5	4	3	3	3	3	4	5	6	8	10	12	16	19

- A. 9
- B. 21
- C. 14
- D. 16

**Q11.** What is the Solar Heat Gain Factor (SHGF) value for window glass (see **Table Q11**) with the condition as below?

*Glass facing west*

*December 21 at 0 deg N lat*

**Table Q11: Maximum Solar Heat Gain Factor (SHGF) for Sunlit Glass**

0 Deg													16 Deg												
N													N												
	NNE/	NE/	ENE/	E/	ESE/	SE/	SSE/	S	HOR		NNE/	NE/	ENE/	E/	ESE/	SE/	SSE/	S	HOR						
Jan.	34	34	88	177	234	254	235	182	118	296	Jun.	30	30	55	147	210	244	251	223	199	248				
Feb.	36	39	132	205	245	247	210	141	67	306	Feb.	33	33	96	180	231	247	233	188	154	275				
Mar.	38	87	170	223	242	223	170	87	38	303	Mar.	35	53	140	205	239	235	197	138	93	291				
Apr.	71	134	193	224	221	184	118	38	37	284	Apr.	39	99	172	216	227	204	150	77	45	289				
May	113	164	203	218	201	154	80	37	37	265	May	52	132	189	218	215	179	115	45	41	282				
June	129	173	206	212	191	140	66	37	37	255	June	66	142	194	217	207	167	99	41	41	277				
July	115	164	201	213	195	149	77	38	38	260	July	55	132	187	214	210	174	111	44	42	277				
Aug.	75	134	187	216	212	175	112	39	38	276	Aug.	41	100	168	209	219	196	143	74	46	282				
Sep.	40	84	163	213	231	213	163	84	40	293	Sep.	36	50	134	196	227	224	191	134	93	282				
Oct.	37	40	129	199	236	238	202	135	66	299	Oct.	33	33	95	174	223	237	225	183	150	270				
Nov.	35	35	88	175	230	250	230	179	117	293	Nov.	30	30	55	145	206	241	247	220	196	246				
Dec.	34	34	71	164	226	253	240	196	135	288	Dec.	29	29	41	132	198	241	254	233	212	234				

4 Deg													20 Deg												
N													N												
	NNE/	NE/	ENE/	E/	ESE/	SE/	SSE/	S	HOR		NNE/	NE/	ENE/	E/	ESE/	SE/	SSE/	S	HOR						
Jan.	33	33	79	170	229	252	237	193	141	286	Jan.	29	29	48	138	201	243	253	233	214	232				
Feb.	35	35	123	190	242	248	215	152	88	301	Feb.	31	31	88	173	226	244	238	201	174	263				
Mar.	38	77	163	219	242	227	177	96	43	302	Mar.	34	49	132	200	237	236	206	152	115	284				
Apr.	55	125	189	223	223	190	126	43	38	287	Apr.	38	92	166	213	228	208	158	91	58	287				
May	93	154	200	220	206	161	89	38	38	272	May	47	123	184	217	217	184	124	54	42	283				
June	110	164	202	215	196	147	73	38	38	263	June	59	135	189	216	210	173	108	45	42	279				
July	96	154	197	215	200	156	85	39	38	267	July	48	124	182	213	212	179	119	53	43	278				
Aug.	59	124	184	215	214	181	120	42	40	279	Aug.	40	91	162	206	220	200	152	88	57	280				
Sep.	39	75	156	209	231	216	170	93	44	293	Sep.	36	46	127	191	225	225	199	148	114	275				
Oct.	36	36	120	193	234	239	207	148	86	294	Oct.	32	32	87	167	217	236	231	196	170	258				
Nov.	34	34	79	168	226	248	232	190	139	284	Nov.	29	29	48	136	197	239	249	229	211	230				
Dec.	33	33	62	157	221	250	242	206	160	277	Dec.	27	27	35	122	187	238	254	241	226	217				

- A. 187
- B. 198
- C. 226
- D. 221



**Q12.** What is the Shading Coefficient (SC) value for window glass (see **Table Q12**) with the condition as below?

*Sunlit double insulating glass window*  
*Heat absorbing gray-tinted outer glass (1/4 in.)*  
*3/16 in. air space*  
*Clear inner glass (1/4 in.)*  
*Unshaded outside & inside ( $h_o = 3.0$ )*

**Table Q12: Shading Coefficients for Glass without or with Interior shading.**

	Types of Glass	Nominal Thickness each Light	No Interior Shading		Types of Interior Shading				
			$h_o = 4.0$	$h_o = 3.0$	Venetian Blinds		Roller Shades		
					Medium	Light	Dark	Opaque	Translucent
SINGLE GLASS	Single								
	Clear	3/32 to 1/4	1.00	1.00					
	Clear	1/4 to 1/2	0.94	0.95					
	Clear	3/8	0.90	0.99	0.64	0.55	0.59	0.25	0.39
	Clear	1/2	0.87	0.88					
	Clear Pattern	1/8 to 9/32	0.83	0.85					
	Heat Absorbing Pattern	1/8	0.83	0.85					
	Heat Absorbing	3/16 to 1/4	0.69	0.73					
	Heat Absorbing Pattern	3/16 to 1/4	0.69	0.73	0.57	0.53	0.45	0.30	0.36
	Tinted	1/8 to 7/32	0.69	0.73					
Heat Absorbing or Pattern		0.60	0.64	0.54	0.52	0.40	0.28	0.32	
Heat Absorbing	3/8	0.60	0.64						
INSULATING GLASS	Double								
	Clear Out	3/32, 1/8	0.88	0.88	0.57	0.51	0.60	0.25	0.37
	Clear In								
	Clear Out	1/4	0.81	0.82					
	Clear In								
	Heat Absorbing Out	1/4	0.55	0.58					
	Clear In								
Triple									
Clear	1/4	0.71							
Clear	1/8	0.80							

- A. 0.73
- B. 0.69
- C. 0.55
- D. 0.58

**Q13.** Given the following design conditions, determine the cooling load at 1200 hours due to lighting within the space:

- Lights on from 8.00 am until 6.00 pm
- Two (2) 40-W lamps per fixture, 12 fixtures with only 9 on at a time
- Ballast factor = 1.20
- CLF = 0.72

- A. 88 Btu/hr
- B. 311 Btu/hr
- C. 2122 Btu/hr
- D. 2946 Btu/hr



**Q14.** Determine the total cooling load at 1200 hours due to 6 people in an office from 9.00 am to 5.00 pm based on the data given and referring to **Table Q14**. The office temperature is 78°F.

$$Q_s/\text{person} = 230 \text{ Btu/hr}$$

$$Q_L/\text{person} = 190 \text{ Btu/hr}$$

**Table Q14: Sensible heat cooling load factors for people**

Total hours in space	Hours after people enter space											
	1	2	3	4	5	6	7	8	9	10	11	12
2	0.65	0.74	0.16	0.11	0.08	0.06	0.05	0.04	0.03	0.02	0.02	0.01
4	0.65	0.75	0.81	0.85	0.24	0.17	0.13	0.10	0.07	0.06	0.04	0.03
6	0.65	0.75	0.81	0.85	0.89	0.91	0.29	0.20	0.15	0.12	0.09	0.07
8	0.65	0.75	0.81	0.85	0.89	0.91	0.93	0.95	0.31	0.22	0.17	0.13
10	0.65	0.75	0.81	0.85	0.89	0.91	0.93	0.95	0.96	0.97	0.33	0.24

- A. 1140 Btu
- B. 1380 Btu
- C. 2280 Btu
- D. 2520 Btu

**Q15.** Which of the following is not one of the three methods used to size duct systems?

- A. static gain
- B. equal friction
- C. equal freedom of flow
- D. equal velocity



**Q16.** Typical duct velocities for low-velocity systems are shown in **Table Q16**. What is the airflow of the main duct supply for hospital bedrooms in feet per minute?

**Table Q16: Suggested duct velocities for a low-velocity duct system, ft/min (m/s)**

Application	Main ducts		Branch ducts	
	Supply	Return	Supply	Return
Residences	1,000 (5.1)	800 (4.1)	600 (3)	600 (3)
Apartments	1,500 (7.6)	1,300 (6.6)	1,200 (6.1)	1,000 (5.1)
Hotel bedrooms				
Hospital bedrooms				
Private offices	1,800 (9.1)	1,400 (7.1)	1,400 (7.1)	1,200 (6.1)
Director's rooms				
Libraries				
Theaters	1,300 (6.6)	1,100 (5.6)	1,000 (5.1)	800 (4.1)
Auditoriums				
General offices	2,000 (10.2)	1,500 (7.6)	1,600 (8.1)	1,200 (6.1)
Expensive restaurants				
Expensive stores				
Banks	2,000 (10.2)	1,500 (7.6)	1,600 (8.1)	1,200 (6.1)
Average stores				
Cafeterias				
Industrial	2,500 (12.7)	1,800 (9.1)	2,200 (11.2)	1,600 (8.1)

- A. 1000
- B. 1200
- C. 1300
- D. 1500

**Q17.** What does it indicate by equal friction in a duct system?

- A. this method of sizing supply duct systems figures constant pressure loss per unit length
- B. this method of sizing return duct systems figures constant pressure loss per unit length
- C. this method of design is computer-operated
- D. this describes the unit of measure in which airflow is gauged

**Q18.** Which of the following is how the duct sizes on the air friction chart are expressed?

- A. Square
- B. Round
- C. Oval
- D. Rectangular





- Q19.** The average velocity of a 20-in round duct is 85 fpm. What is the airflow (cfm) in the duct?
- A. 112
  - B. 152
  - C. 171
  - D. 185
- Q20.** With no shading, a room has a total window area of 100 ft<sup>2</sup>. The temperature in the room is 80°F, and the daily average temperature outside is 84°F. At 4 pm solar time, what is the cooling load from conduction heat gain through the glass?
- Note: This window features a single pane of glass and an aluminum sash with a thermal break ( $U = 0.81 \text{ Btu} / (\text{hr} \cdot \text{ft} \cdot \text{F})$ )
- A. 324 Btu/hr
  - B. 400 Btu/hr
  - C. 64,800 Btu/hr
  - D. 80,000 Btu/hr

**SECTION B (80 marks)**

- Q21** (a) According to ASHRAE (2013), what is the definition of thermal comfort? (2 marks)
- (b) Describe four (4) of the environmental and human-related characteristics that can impact the thermal comfort of building occupants. (8 marks)
- (c) You intended to move to a new apartment flat and have shortlisted 3 units as indicated (A, B and C) in **Figure Q21(c)**. Which of the three units will be the worst choice in terms of thermal comfort? State your reasons. (5 marks)
- (d) Calculate the winter U-factor for a wall made up of 4 inch (100 mm) face brick ( $K = 1.1 \text{ W/m K}$ ), 4 inch (100 mm) common brick ( $K = 0.6 \text{ W/m K}$ ), and 1/2 inch (13 mm) gypsum plaster ( $K = 0.11 \text{ W/m K}$ ). (3 marks)

**Q22** (a) Referring to the following information, **Figure Q22 (a)**, **Table Q22 (a-i)**, and **Table Q22 (a-ii)** calculate the heat gain by conduction through an east-facing wall at Hour 12 in July for Room 102 located at Kuala Lumpur (3.1390° N, 101.6869° E) which is a new building ( $K=0.83$ ).

- Group D walls
- Floor-to-ceiling height = 12.14 ft
- U-factor of the wall =  $0.284 \text{ Btu}/(\text{hr} \cdot \text{ft}^2 \cdot ^\circ\text{F})$
- Indoor dry-bulb temperature =  $78^\circ\text{F}$
- Outdoor dry-bulb temperature =  $95^\circ\text{F}$

(5 marks)

(b) Referring to the following information, **Figure Q22 (a)**, **Figure Q22 (b)**, **Table Q22 (b-i)**, **Table Q22 (b-ii)**, and **Table Q22 (b-iii)**, calculate the heat gain by solar radiation through the windows on a west-facing wall at Hour 14 in July for Room 101 at Kuala Lumpur (3.1390° N, 101.6869° E).

- Eight (8) single-layer clear windows with interior shading light roller type. Each window is 3.94ft wide x 4.92ft high.

(5 marks)

(c) Referring to the following information, calculate both the sensible and latent heat gains due to air infiltration from outdoors for Room 101.

- Area of floor =  $247.57 \text{ ft}^2$
- Floor-to-ceiling height = 12.14 ft
- Amount of infiltration = 0.76 air change/hour
- Indoor conditions =  $75^\circ\text{F}$  dry-bulb temperature and 0.009 specific humidity
- Outdoor conditions =  $100^\circ\text{F}$  dry-bulb temperature and 0.02 specific humidity

(7 marks)

(d) Referring to the following information, **Table Q22 (d-i)** and **Table Q22 (d-ii)** calculate the internal heat gains from people and lighting.

- Number of people = 10 with peoples participating in moderately active office work. CLF = 0.84
- Only 8 lamps were used from 10 lamps.
- Working hours = 8 hours
- a classification = 0.55
- b classification = C
- Designing time = 16
- Amount of lighting in space = 1.1 watt/ft<sup>2</sup>
- Area of floor = 247.57 ft<sup>2</sup>
- Ballast factor = 1.3 (fluorescent lights)

(7 marks)

**Q23** (a) Given the following components of cooling load in **Table Q23(a)**:

**Table Q23(a)**

	Sensible heat gain (W)	Latent heat gain (W)
Conduction through roof	2050	
Conduction through exterior walls	200	
Conduction through windows	400	
Solar radiation through windows or skylights	5000	
Heat gain from people	2000	950
Heat gain from lights	6500	
Heat gain from office equipment's	2500	
Air infiltration through cracks from outdoors	800	650
Cooling load due to ventilation brought in by the central HVAC system	1850	1900
Heat gains from the supply fan	1000	

i. Calculate the sensible heat ratio for the space?

(4 marks)

ii. Assuming the air is supplied to the space at 13°C dry-bulb and the space dry-bulb temperature is 25°C, calculate the quantity of air must be supplied to the space?

(5 marks)

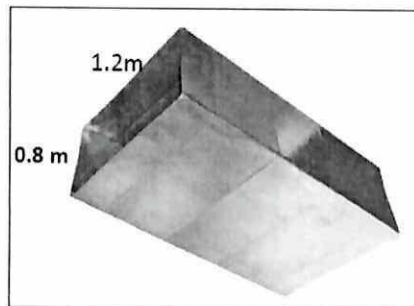
- (b) The following sensible heat gains exist for three spaces being served by the same centralized HVAC systems are shown in **Table Q23(b)**;

**Table Q23(b)**

	<b>Total space sensible heat gain (W)</b>	
	<b>9.00 am</b>	<b>4.00 pm</b>
Room 201	20,500	6,000
Room 202	10,000	25,000
Room 203	24,500	6,000

- i. Explain when does the block load occurs? (4 marks)
  
- ii. Calculate the block space sensible load? (2 marks)
  
- iii. Suppose that you are sizing a constant-volume supply fan. Calculate the "sum-of-peaks" space sensible load? (2 marks)

- Q24** (a) Calculate the aspect ratio using the rectangular air duct dimensions shown in Figure Q24(a). Explain why this rectangular duct's capital and operating cost is higher than a square duct of the same flow area.



**Figure Q24(a)**

(5 marks)

- (b) Explain why rectangular ducts are still commonly used in buildings despite the higher capital and operating cost.

(4 marks)

- (c) The following **Figure Q24 (c)** shows a typical duct layout.

- i. Using the equal friction method and friction chart in **Figure Q24 (c-i)**, determine the size of each ducting (A – I) and duct loss in each ducting if using round ducts with the maximum velocity of air allowed in the system as 5 m/s. You also need to fill the **Table Q24(c)**:

**Table Q24(c)**

ID	TYPE	Volume flow rate (m <sup>3</sup> /s)	ΔP (Pa/m)	Velocity (m/s)	Diameter (m)	Length (m)	Duct loss (Pa)
A	Duct						
B	Tee						
C	Branch						
D	Duct						
E	Tee						
F	Branch						
G	Branch						
H	90 Bend						
I	Branch						

- ii. Based on the complete **Table Q24(c)**, determine the index run on the system.

(12 marks)

- END OF QUESTIONS -

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Figure Q21 (c)

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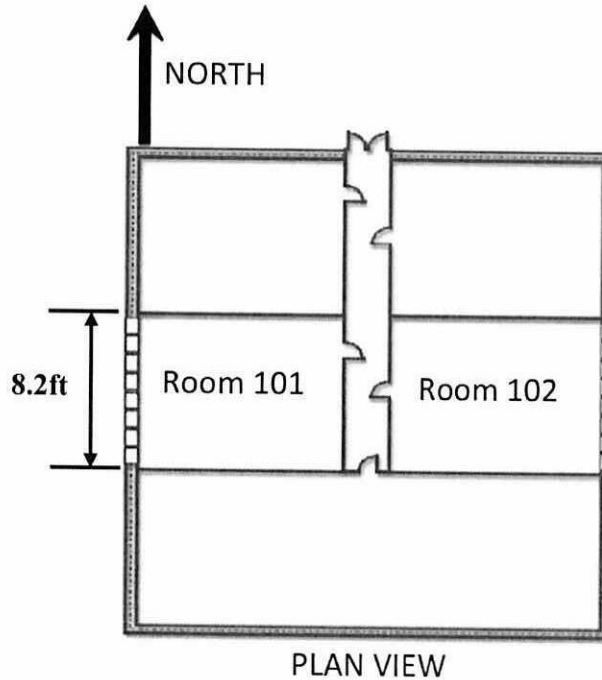


Figure Q22 (a)

Table Q22 (a-i): Cooling Load Temperature Differences for Calculating Cooling Load from Sunlit Walls

North Latitude Wall Facing	Solar Time, hr																								Hr of Maxi- mum CLTD	Mini- mum CLTD	Maxi- mum CLTD	Differ- ence CLTD	
	1	2	3	4	5	6	7	8	9	10	11	12	13	14	15	16	17	18	19	20	21	22	23	24					
Group A Walls																													
N	14	14	14	13	13	13	12	12	11	11	10	10	10	10	10	10	11	11	12	12	13	14	14	2	10	14	4		
NE	19	19	19	18	17	17	16	15	15	15	15	15	16	16	17	18	18	18	19	19	20	20	20	22	15	20	5		
E	24	24	23	23	22	21	20	19	19	18	18	18	18	18	19	20	21	22	23	24	24	25	25	25	22	18	25	7	
SE	24	23	23	22	21	20	20	19	18	18	18	18	18	19	20	21	22	23	23	24	24	24	24	24	22	18	24	6	
S	20	20	19	19	18	18	17	16	16	15	14	14	14	14	14	15	16	17	18	19	19	20	20	20	23	14	20	6	
SW	25	25	25	24	24	23	22	21	20	19	19	18	17	17	17	17	18	19	20	22	23	24	25	24	17	25	8		
W	27	27	26	26	25	24	24	23	22	21	20	19	19	18	18	18	19	20	22	23	25	26	26	1	18	27	9		
NW	21	21	21	20	20	19	19	18	17	16	16	15	15	14	14	14	15	16	17	18	19	20	21	14	21	7	7		
Group B Walls																													
N	15	14	14	13	12	11	11	10	9	9	9	9	9	9	10	11	12	13	14	14	15	15	15	24	8	15	7		
NE	19	18	17	16	15	14	13	12	12	13	14	15	16	17	18	19	20	20	21	21	21	20	20	21	12	21	9		
E	23	22	21	20	18	17	16	15	15	15	17	19	21	22	24	25	26	26	27	27	26	26	25	24	20	15	27	12	
SE	23	22	21	20	18	17	16	15	14	14	15	16	18	20	21	23	24	25	26	26	26	26	25	24	21	14	26	12	
S	21	20	19	18	17	15	14	13	12	11	11	11	11	12	14	15	17	19	20	21	22	22	22	21	11	22	11		
SW	27	26	25	24	22	21	19	18	16	15	14	14	13	13	14	15	17	20	22	25	27	28	28	24	13	28	15		
W	29	28	27	26	24	23	21	19	18	17	16	15	14	14	15	17	19	22	25	27	29	30	30	24	14	30	16		
NW	23	22	21	20	19	18	17	15	14	13	12	12	12	11	12	12	13	15	17	19	21	22	23	24	11	23	9		
Group C Walls																													
N	15	14	13	12	11	10	9	8	8	7	7	7	8	9	10	12	13	14	15	16	17	17	17	16	7	17	10		
NE	19	17	16	14	13	11	10	10	11	13	13	13	17	19	20	21	22	22	23	23	23	23	22	21	20	20	23	13	
E	22	21	19	17	15	14	12	12	12	13	16	19	22	25	27	29	29	30	30	30	29	28	27	26	24	18	12	30	18
SE	22	21	19	17	15	14	12	12	12	13	16	19	22	24	26	28	28	29	29	29	28	27	26	24	19	12	29	17	
S	21	19	18	16	15	13	12	10	9	9	9	10	11	14	17	20	22	24	25	26	25	25	24	22	20	9	26	17	
SW	29	27	25	22	20	18	16	15	13	12	11	11	11	13	15	18	22	26	29	32	33	33	32	31	22	11	33	22	
W	31	29	27	25	22	20	18	16	14	13	12	12	12	13	14	16	20	24	29	32	35	35	35	31	22	12	35	23	
NW	25	23	21	20	18	16	14	13	11	10	10	10	10	11	12	13	15	18	22	25	27	27	26	22	10	27	17		
Group D Walls																													
N	15	13	12	10	9	7	6	6	6	6	6	6	7	8	10	12	13	15	17	18	19	19	19	18	16	21	6	19	13
NE	17	15	13	11	10	8	7	8	10	14	17	20	22	23	23	24	24	25	25	24	23	22	20	18	19	7	25	18	
E	19	17	15	13	11	9	8	9	12	17	22	27	30	32	33	33	32	32	31	30	28	26	24	22	16	8	33	25	
SE	20	17	15	13	11	10	8	8	10	13	17	22	26	29	31	32	32	32	31	30	28	26	24	22	17	8	32	24	
S	19	17	15	13	11	9	8	7	6	6	6	7	9	12	16	20	24	27	29	29	29	27	26	24	22	19	6	29	23
SW	28	25	22	19	16	14	12	10	9	8	8	8	10	12	16	21	27	32	36	38	38	37	34	31	21	8	38	30	
W	31	27	24	21	18	15	13	11	10	9	9	9	10	11	14	18	24	30	36	40	41	40	38	34	21	9	41	32	
NW	25	22	19	17	14	12	10	9	8	7	7	7	8	9	10	12	14	18	22	27	31	32	32	30	27	22	7	32	25



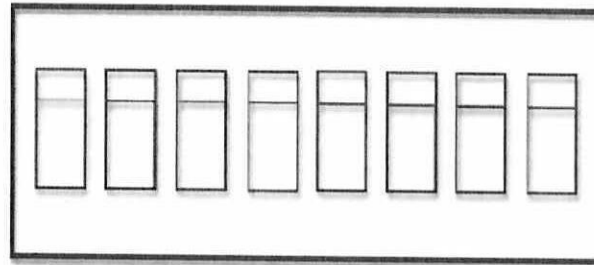
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Table Q22 (a-ii): CLTD Correction for Latitude and Month Applied to Walls and Roofs, North Latitudes.

**Table 3.12 CLTD Correction For Latitude and Month Applied to Walls and Roofs, North Latitudes**

Lat.	Month	N	NNE NNW	NE NW	ENE WNW	E W	ESE WSW	SE SW	SSE SSW	S	HOR
0	Dec	-3	-5	-5	-5	-2	0	3	6	9	-1
	Jan/Nov	-3	-5	-4	-4	-1	0	2	4	7	-1
	Feb/Oct	-3	-2	-2	-2	-1	-1	0	-1	0	0
	Mar/Sept	-3	0	1	-1	-1	-3	-3	-5	-8	0
	Apr/Aug	5	4	3	0	-2	-5	-6	-8	-8	-2
	May/Jul	10	7	5	0	-3	-7	-8	-9	-8	-4
	Jun	12	9	5	0	-3	-7	-9	-10	-8	-5
8	Dec	-4	-6	-6	-6	-3	0	4	8	12	-5
	Jan/Nov	-3	-5	-6	-5	-2	0	3	6	10	-4
	Feb/Oct	-3	-4	-3	-3	-1	-1	1	2	4	-1
	Mar/Sept	-3	-2	-1	-1	-1	-2	-2	-3	-4	0
	Apr/Aug	2	2	2	0	-1	-4	-5	-7	-7	-1
	May/Jul	7	5	4	0	-2	-5	-7	-9	-7	-2
	Jun	9	6	4	0	-2	-6	-8	-9	-7	-2
16	Dec	-4	-6	-8	-8	-4	-1	4	9	13	-9
	Jan/Nov	-4	-6	-7	-7	-4	-1	4	8	12	-7
	Feb/Oct	-3	-5	-5	-4	-2	0	2	5	7	-4
	Mar/Sept	-3	-3	-2	-2	-1	-1	0	0	0	-1
	Apr/Aug	-1	0	-1	-1	-1	-3	-3	-5	-6	0
	May/Jul	4	3	3	0	-1	-4	-5	-7	-7	0
	Jun	6	4	4	1	-1	-4	-6	-8	-7	0



Elevation view (Room 101)

Figure Q22 (b)





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Table Q22 (b-i): Shading coefficient for glass without or with interior shading by venetian blinds or roller shades.

	Type of Glass	Nominal Thickness Each Light <sup>a</sup>	Solar Trans. <sup>b</sup>	No Interior Shading		Type of Interior Shading				
				<i>h<sub>i</sub></i> = 4.0	<i>h<sub>i</sub></i> = 3.0	Venetian Blinds		Roller Shades		
						Medium	Light	Opaque		Translucent
						Dark	Light			
SINGLE GLASS	Single Clear	3/32 to 1/4	0.87-0.80	1.00	1.00					
	Single Clear	1/4 to 1/2	0.80-0.71	0.94	0.95					
	Single Clear	3/8	0.72	0.90	0.92	0.64	0.55	0.59	0.25	0.39
	Single Clear	1/2	0.67	0.87	0.88					
	Single Clear Pattern	1/8 to 9/32	0.87-0.79	0.83	0.85					
	Single Heat Absorbing Pattern	1/8		0.83	0.85					
	Single Heat Absorbing <sup>c</sup>	3/16 to 1/4	0.46	0.69	0.73					
	Single Heat Absorbing Pattern	3/16 to 1/4		0.69	0.73	0.57	0.53	0.45	0.30	0.36
	Single Tinted	1/8 to 7/32	0.59-0.45	0.69	0.73					
	Single Heat Absorbing or Pattern		0.44-0.30	0.60	0.64	0.54	0.52	0.40	0.28	0.32
	Single Heat Absorbing <sup>c</sup>	3/8	0.34	0.60	0.64					
	Single Heat Absorbing or Pattern	1/2	0.44-0.30	0.53	0.58	0.42	0.40	0.36	0.28	0.31
	Single Reflective Coated Glass		0.24	0.30	0.40					
			0.40	0.50	0.25	0.23				
			0.50	0.60	0.33	0.29				
			0.50	0.60	0.42	0.38				
			0.50	0.60	0.50	0.44				
INSULATING GLASS	Double <sup>d</sup> Clear Out	3/32, 1/8	0.71 <sup>a</sup>	0.88	0.88	0.57	0.51	0.60	0.25	0.37
	Double Clear In		0.61 <sup>a</sup>	0.81	0.82					
	Double Clear Out	1/4								
	Double Heat Absorbing Out	1/4	0.36 <sup>a</sup>	0.55	0.58					
	Double Clear In					0.39	0.36	0.40	0.22	0.30
	Double Reflective Coated Glass			0.20	0.30					
				0.30	0.40					
Triple Clear	1/4		0.71							
Triple Clear	1/8		0.80							

Table Q22 (b-ii): Maximum SHGF for Sunlit Glass (0° North Latitude), *Btu/(hr · ft<sup>2</sup>)*

0 Deg											
	N	NNE/ NNW	NE/ NW	ENE/ WNW	E/ W	ESE/ WSW	SE/ SW	SSE/ SSW	S	HOR	
Jan.	34	34	88	177	234	254	235	182	118	296	
Feb.	36	39	132	205	245	247	210	141	67	306	
Mar.	38	87	170	223	242	223	170	87	38	303	
Apr.	71	134	193	224	221	184	118	38	37	284	
May	113	164	203	218	201	154	80	37	37	265	
June	129	173	206	212	191	140	66	37	37	255	
July	115	164	201	213	195	149	77	38	38	260	
Aug.	75	134	187	216	212	175	112	39	38	276	
Sep.	40	84	163	213	231	213	163	84	40	293	
Oct.	37	40	129	199	236	238	202	135	66	299	
Nov.	35	35	88	175	230	250	230	179	117	293	
Dec.	34	34	71	164	226	253	240	196	138	288	



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**Table Q22 (b-iii): Cooling Load Factors for Glass with Interior Shading, North Latitude (All Room Construction)**

Fenes- stration Facing	Solar Time, hr																							
	1	2	3	4	5	6	7	8	9	10	11	12	13	14	15	16	17	18	19	20	21	22	23	24
N	0.08	0.07	0.06	0.06	0.07	0.73	0.66	0.65	0.73	0.80	0.86	0.89	0.89	0.86	0.82	0.75	0.78	0.91	0.24	0.18	0.15	0.13	0.11	0.10
NNE	0.03	0.03	0.02	0.02	0.03	0.64	0.77	0.62	0.42	0.37	0.37	0.37	0.36	0.35	0.32	0.28	0.23	0.17	0.08	0.07	0.06	0.05	0.04	0.04
NE	0.03	0.02	0.02	0.02	0.02	0.56	0.76	0.74	0.58	0.37	0.29	0.27	0.26	0.24	0.22	0.20	0.16	0.12	0.06	0.05	0.04	0.04	0.03	0.03
ENE	0.03	0.02	0.02	0.02	0.02	0.52	0.76	0.80	0.71	0.52	0.31	0.26	0.24	0.22	0.20	0.18	0.15	0.11	0.06	0.05	0.04	0.04	0.03	0.03
E	0.03	0.02	0.02	0.02	0.02	0.47	0.72	0.80	0.76	0.62	0.41	0.27	0.24	0.22	0.20	0.17	0.14	0.11	0.06	0.05	0.05	0.04	0.03	0.03
ESE	0.03	0.03	0.02	0.02	0.02	0.41	0.67	0.79	0.80	0.72	0.54	0.34	0.27	0.24	0.21	0.19	0.15	0.12	0.07	0.06	0.05	0.04	0.04	0.03
SE	0.03	0.03	0.02	0.02	0.02	0.30	0.57	0.74	0.81	0.79	0.68	0.49	0.33	0.28	0.25	0.22	0.18	0.13	0.08	0.07	0.06	0.05	0.04	0.04
SSE	0.04	0.03	0.03	0.03	0.02	0.12	0.31	0.54	0.72	0.81	0.81	0.71	0.54	0.38	0.32	0.27	0.22	0.16	0.09	0.08	0.07	0.06	0.05	0.04
S	0.04	0.04	0.03	0.03	0.03	0.09	0.16	0.23	0.38	0.58	0.75	0.83	0.80	0.68	0.50	0.35	0.27	0.19	0.11	0.09	0.08	0.07	0.06	0.05
SSW	0.05	0.04	0.04	0.03	0.03	0.09	0.14	0.18	0.22	0.27	0.43	0.63	0.78	0.84	0.80	0.66	0.46	0.25	0.13	0.11	0.09	0.08	0.07	0.06
SW	0.05	0.05	0.04	0.04	0.03	0.07	0.11	0.14	0.16	0.19	0.22	0.38	0.59	0.75	0.83	0.81	0.69	0.45	0.16	0.12	0.10	0.09	0.07	0.06
WSW	0.05	0.05	0.04	0.04	0.03	0.07	0.10	0.12	0.14	0.16	0.17	0.23	0.44	0.64	0.78	0.84	0.78	0.55	0.16	0.12	0.10	0.09	0.07	0.06
W	0.05	0.05	0.04	0.04	0.03	0.06	0.09	0.11	0.13	0.15	0.16	0.17	0.31	0.53	0.72	0.82	0.81	0.61	0.16	0.12	0.10	0.08	0.07	0.06
WNW	0.05	0.05	0.04	0.03	0.03	0.07	0.10	0.12	0.14	0.16	0.17	0.18	0.22	0.43	0.65	0.80	0.84	0.66	0.16	0.12	0.10	0.08	0.07	0.06
NW	0.05	0.04	0.04	0.03	0.03	0.07	0.11	0.14	0.17	0.19	0.20	0.21	0.22	0.30	0.52	0.73	0.82	0.69	0.16	0.12	0.10	0.08	0.07	0.06
NNW	0.05	0.05	0.04	0.03	0.03	0.11	0.17	0.22	0.26	0.30	0.32	0.33	0.34	0.34	0.39	0.61	0.82	0.76	0.17	0.12	0.10	0.08	0.07	0.06
HOR.	0.06	0.05	0.04	0.04	0.03	0.12	0.27	0.44	0.59	0.72	0.81	0.85	0.85	0.81	0.71	0.58	0.42	0.25	0.14	0.12	0.10	0.08	0.07	0.06

**Table Q22 (d-i): Cooling Load Factors when Lights are on for 8 Hours**

"a" Class- ification	"b" Class- ification	Number of hours after lights are turned on																							
		0	1	2	3	4	5	6	7	8	9	10	11	12	13	14	15	16	17	18	19	20	21	22	23
0.45	A	0.02	0.46	0.57	0.65	0.72	0.77	0.82	0.85	0.88	0.46	0.37	0.30	0.24	0.19	0.15	0.12	0.10	0.08	0.06	0.05	0.04	0.03	0.03	0.02
	B	0.07	0.51	0.56	0.61	0.65	0.68	0.71	0.74	0.77	0.34	0.31	0.28	0.25	0.22	0.20	0.18	0.16	0.15	0.13	0.12	0.11	0.10	0.09	0.08
	C	0.11	0.55	0.58	0.60	0.63	0.65	0.67	0.69	0.71	0.28	0.26	0.25	0.23	0.22	0.20	0.19	0.18	0.17	0.16	0.15	0.14	0.13	0.12	0.12
	D	0.14	0.58	0.60	0.61	0.62	0.63	0.64	0.65	0.66	0.22	0.22	0.21	0.20	0.20	0.19	0.19	0.18	0.18	0.17	0.16	0.16	0.16	0.16	0.15
0.55	A	0.01	0.56	0.65	0.72	0.77	0.82	0.85	0.88	0.90	0.37	0.30	0.24	0.19	0.16	0.13	0.10	0.08	0.07	0.05	0.04	0.03	0.03	0.02	0.02
	B	0.06	0.60	0.64	0.68	0.71	0.74	0.76	0.79	0.81	0.28	0.25	0.23	0.20	0.18	0.16	0.15	0.13	0.12	0.11	0.10	0.09	0.08	0.07	0.06
	C	0.09	0.63	0.66	0.68	0.70	0.71	0.73	0.75	0.76	0.23	0.21	0.20	0.19	0.18	0.17	0.16	0.15	0.14	0.13	0.12	0.11	0.11	0.10	0.10
	D	0.11	0.66	0.67	0.68	0.69	0.70	0.71	0.72	0.72	0.18	0.18	0.17	0.17	0.16	0.16	0.15	0.15	0.14	0.14	0.13	0.13	0.13	0.12	0.12
0.65	A	0.01	0.66	0.73	0.78	0.82	0.86	0.88	0.91	0.93	0.29	0.23	0.19	0.15	0.12	0.10	0.08	0.06	0.05	0.04	0.03	0.03	0.02	0.02	0.01
	B	0.04	0.69	0.72	0.75	0.77	0.80	0.82	0.84	0.85	0.22	0.19	0.18	0.16	0.14	0.13	0.12	0.10	0.09	0.08	0.08	0.07	0.06	0.06	0.05
	C	0.07	0.72	0.73	0.75	0.76	0.78	0.79	0.80	0.82	0.18	0.17	0.16	0.15	0.14	0.13	0.12	0.11	0.11	0.10	0.10	0.09	0.08	0.08	0.07
	D	0.09	0.73	0.74	0.75	0.76	0.77	0.77	0.78	0.79	0.14	0.14	0.13	0.13	0.13	0.12	0.12	0.11	0.11	0.11	0.10	0.10	0.10	0.10	0.10
0.75	A	0.01	0.76	0.80	0.84	0.87	0.90	0.92	0.93	0.95	0.21	0.17	0.13	0.11	0.09	0.07	0.06	0.05	0.04	0.03	0.02	0.02	0.02	0.01	0.01
	B	0.03	0.78	0.80	0.82	0.84	0.85	0.87	0.88	0.89	0.15	0.14	0.13	0.11	0.10	0.09	0.08	0.07	0.07	0.06	0.05	0.05	0.04	0.04	0.04
	C	0.05	0.80	0.81	0.82	0.83	0.84	0.85	0.86	0.87	0.13	0.12	0.11	0.10	0.10	0.09	0.09	0.08	0.08	0.08	0.07	0.07	0.06	0.06	0.05
	D	0.06	0.81	0.82	0.82	0.83	0.83	0.84	0.84	0.85	0.10	0.10	0.10	0.09	0.09	0.09	0.08	0.08	0.08	0.08	0.07	0.07	0.07	0.07	0.07

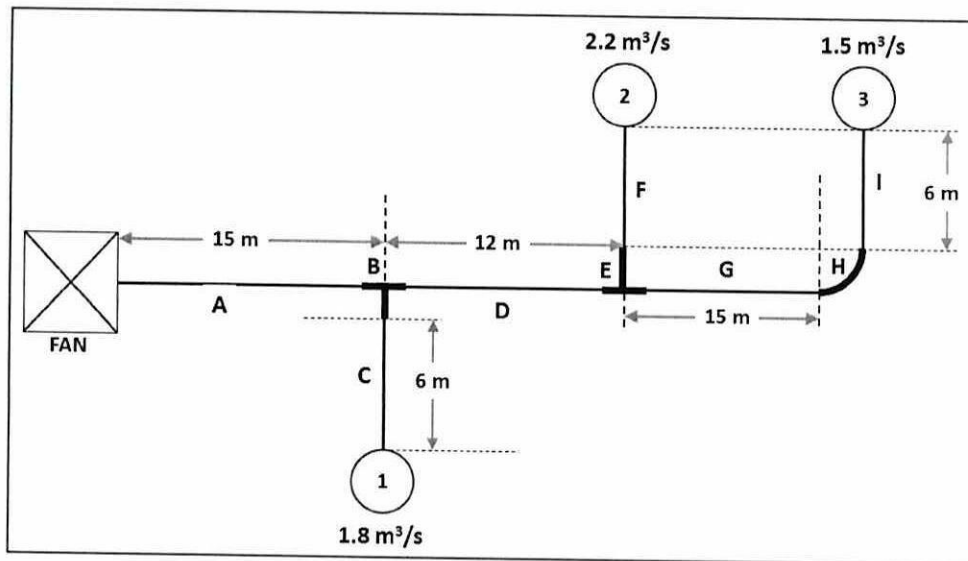


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**Table Q22 (d-ii): Heat Generated by People**

Level of Activity	Sensible Heat Gain (Btu/h)	Latent Heat Gain (Btu/h)
Moderately active work (Office)	256	188
Standing, light work, walking (Store)	256	188
Light bench work (Factory)	273	477
Heavy work (Factory)	580	870
Exercise (Gymnasium)	716	1075



**Figure Q24 (c)**



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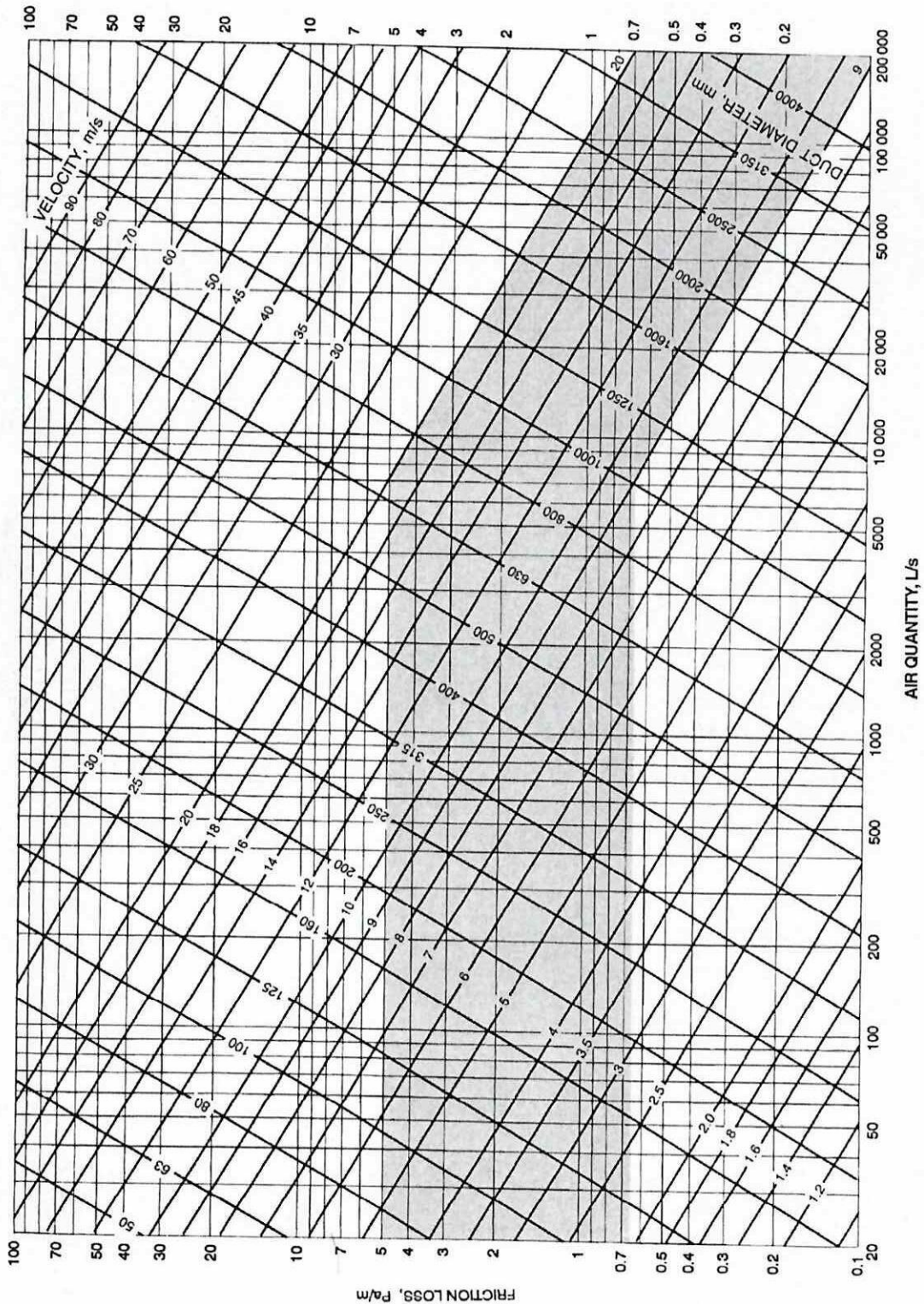


Figure Q24 (c-i)



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### LIST OF FORMULA

#### COOLING LOAD CALCULATION

$$U = 1/R_{\text{total}}$$

$$Q = U \times A \times CLTD$$

$$CLTD \text{ correction: } = (CLTD + LM) \times K + ((78 - Tr) + (To - 80))$$

$$Q_{\text{solar}} = A \times SC \times SHGF \times CLF$$

$$Q_{\text{sensible}} = \text{No. of people} \times \text{sensible heat gain per person} \times CLF$$

$$Q_{\text{latent}} = \text{No. of people} \times \text{latent heat gain per person}$$

Lighting load:

$$Q = qi \times 3.41 \times Fa \times Fs \times CLF$$

Heat gain from infiltration;

$$Q_s = 1.1 \times \Delta T \times CFM$$

$$QL = 4840 \times \Delta \omega \times CFM$$

$$CFM = \text{volume} + Na/60$$

$$\text{Supply airflow} = \text{sensible heat gain} / [1210 \times (\text{room DB} - \text{supply DB})]$$

#### AIR DUCTING DESIGN

$$\Delta P_{A-B} = \Delta P_{A,f} + \Delta P_{B,f} + \Delta P_{u-b} + \Delta P_{\text{exit}}$$

$$\Delta P_{A,f} = (0.022243 \times \bar{Q}_{\text{air}}^{1.852} \times L_A) / D^{4.973}_A; \quad L = \text{length}, \quad D = \text{diameter}, \quad \bar{Q} = \text{flow rate}$$

$$\Delta P_{u-b} = C_{u-b} (\rho V_d^2 / 2); \quad C = \text{dynamic loss coefficient}, \quad \rho = 1.2, \quad V = \text{velocity}$$

$$\Delta P_{\text{exit}} = C_{\text{exit}} (\rho V_l^2 / 2)$$

